- 1 **Please state you name and address for the record.**
- $\overline{2}$ Bill Smith

3 33 South Easter Island Circle

Englewood FL 34223 $\overline{4}$

5 **What is the purpose of your testimony?**

6 To discuss the type of thermal demand meters at dispute in this case, the TMT Form 6-S $\overline{7}$ Duncan Landis & Gyr meter, my role in helping design the meter, my work history with the 8 manufacturer of the meters in dispute in this docket, my knowledge of the mechanics of how 9 these thermal demand meters work, and what 1 believe caused these meters to overregister 10 demand when tested by FPL. I will also discuss the impact that the sun has on thermal demand 11 meters, the proper way these meters should be calibrated, and, how, in my experience, the 12 percentage of error for meters that over-register is calculated.

13 **Please indicate your educational and professional background.**

14 15 16 17 18 19 20 21 22 23 I graduated from North Vernon Indiana High School in 1947. I then served my country in the United States Navy for nine years where I was an Electronics Technician (ET) and a Nuclear Technician. In 1956, I was accepted into Purdue University where I majored in electrical engineering. I graduated from Purdue with a degree in electrical engineering in January of 1961. In 1958, I went to work for Duncan Landis & Gyr. This is the company that made the meters that are in dispute in this docket. I worked there for around **13** years, until 1972. In 1973 I went to work with Anchor Electric. Anchor manufactured meter mounting device. In 1985, I worked with the Astra Corporation, a company that made and sold metering transformer. Shortly thereafter, I worked for the Utility Test Equipment Company (UTEC). UTEC designs, manufactures and distributes meter test equipment. I later returned to Anchor 24 where I finished my career and retired in 1996.

25

DOCUMENT NUMBER-DATE 07589 JUL 12 3 FPSC-COMMISSION CLERK

1 Have you been involved with meters and meter testing equipment pretty much your 2 whole professional career?

3 Yes.

4 What were your duties and responsibilities when you worked with Duncan Landis & Gyr?

5 6 7 8 9 As an electrical engineer, I had a host of duties that involved the meters and test equipment that the Company manufactured. With respect to thermal demand meters, like the ones involved in this docket, my responsibilities included working on the design of the meters and ensuring that quality control was maintained. I also tested meters, including the thermal demand meters, against a standard meter. Finally, I oversaw the testing of meters, including IO thermal demand meters.

11 Did you gain familiarity with the internal workings of thermal demand meters?

12 13 14 Yes. As I mentioned, part of my job included designing the meters. The Company was always seeking ways to improve the thermal demand meter, and part of my responsibilities was to assist with designing improvements to the meter.

15 Are you aware that the meters in this case all overregistered demand?

16 17 18 Yes I am. I have reviewed the testing reports for the meters. The testing reflects that the meters have overregistered demand. Mr. Brown's testimony details the particulars of the amount by which each meter overregistered.

19 20 21 22 One of the issues in this case relates to proving the point in time in which the meters in dispute first started over-registering demand. In your experience in designing and working with thermal demand meters, are you aware of factors that could cause the TMT Form 6-S Duncan Landis & Gyr meter to gradually overregister demand?

23 24 25 The thermal demand meter is a relatively simple measurement tool with few critical parts. I am not aware and do not believe it likely, based upon my knowledge and experience, for the TMT Form **6-S** Duncan Landis & Gyr meter to gradually overregister demand. It is my

impression, based upon a review of depositions taken in this case; that FPL acknowledges that 1 $\sqrt{2}$ the TMT Form 6-S Duncan Landis & Gyr meter does not have mechanical components that would cause the meter to run fast. $\overline{3}$

Why do you say this? $\overline{4}$

5 In the deposition of Keith Herbster, who has worked for FPL for nearly 31 years, with 6 between 15 to 18 of those years being involved with meters, he was asked questions about what 7 mechanically might cause a KWD or kilowatt demand meter to run fast. He answered, correctly 8 in my view, that other than adjustments, there was nothing he is aware of that would cause the 9 kilowatt demand meter to overregister or run fast. See excerpt of deposition testimony of Keith 10 Herbster, pages 86-87 (attached hereto as Exhibit **A).** Also, Brian Faircloth, who states he has 11 tested around 8,000 thermal demand meters, more than anyone at FPL since he has worked in the 12 meter testing center, states "No" in response to the question, "Are you aware of anything that could make these 1V meters gradually or suddenly read high in the field?" See excerpt of the 13 14 deposition testimony of Brian Faircloth at page 64 (attached hereto as Exhibit B). Jim 15 Teachman, another FPL employee responsible for meter testing, also could not identify anything 16 that would cause a thermal demand meter to gradually overregister demand. See excerpt of 17 deposition testimony of Jim Teachman at page 96 (attached hereto as Exhibit C).

18 **What is the likely cause of a thermal demand meter to overregister?**

19 20 I believe that the most likely reason a thermal demand would overregister or read high is due to error in calibrating the meter prior to placing it into service.

21 **Why?**

22 23 24 25 Again, the structure of the meter is pretty basic. It really does not have mechanical parts that are likely to cause the meter to over-register gradually over time. However, the process of calibrating a meter, which involves human manipulation, can result in calibration errors that can cause the meter to either over-register or under-register if miscalibrated.

$\mathbf{1}$ **Explain how you could properly calibrate a meter with today's technology:** $\overline{2}$ This testing example will apply to a gang thermal board that has been set up to test and 3 calibrate a TMT form 6S, two stator, transformer rated meter. A single-phase source is used for $\overline{4}$ potential voltage in parallel and for current in series. A reference standard of known accuracy is *5* used for comparison to meters being tested ("meters under test"). Preferably the standard would be an electronic auto-ranging meter of the same form and programmed with the appropriate 6 7 thermal response curve as the meters to be tested. 8 **INSPECTION** 9 Inspect the meter for any visible damage that may cause a hazard or unsafe 1. condition if tested. 10 2. 11 Inspect the meter for any sign of tampering. If possible correct the problem. If there are no safety concerns continue. 12 3. 13 **ZERO CHECK and Adjustment** 14 4. Remove the original equipment manufacturer (OEM) canopy. Check the black 15 maximum pointer for proper friction while moving it up-scale away from zero. Replace the OEM 16 canopy with a test cover. 5. Place the meter under test in a test socket with the test canopy (test cover) 17 18 securely in place. 19 6. Apply potential voltage only (voltage to match the meter form and type 120V,

20 21 22 240V, 277V, etc.) for a minimum of 2 hours. The black maximum needle should not be in contact with the red instantaneous needle at any time during this test, nor should any current be applied.

23 24 25 *7.* At the end of two-hours record the zero reading. (AS FOUND) If adjustment is necessary, insert a flat slot screwdriver through the test cover hole corresponding to the zero adjustment on the left side of the meter when facing the meter. If adjusting is necessary, adjust

1 2 3 4 zero as possible. the zero to within the blade edge width of the indicating red needle on the zero scale point. If adjusting upscale, move the red pointer slightly past the zero then back to zero. This will allow for any backlash, which may occur. If adjusting downscale move the red pointer to as close to

5 **FULL-SCALE CALIBRATION**

6 7 8 of1 hour. 8. If the potential voltage has not been interrupted for at least 2-hours, the full-scale calibration procedure can begin. Otherwise the meter should be preheated again for a minimum

9 9. 10 11 12 13 scale. Amperage should be selected that will correspond to at least 75% registration of full-scale reading of the meter under test. (This is so, because the manufacturer has originally calibrated and warranted the accuracy of this meter at 75% of full scale.) In a single-phase series test this will correspond to $\frac{3}{4}$ of the amperage needed to reach the desired test point on the full-

14 15 16 10. The selected amperage is applied to the circuit that contains the meters under test as well as the reference standard of known accuracy. The black maximum pointer is moved back to a position that will make contact with the red pointer while testing.

17 18 11. The applied amperage and voltage should be monitored closely to maintain their values within 2% of desired test point. This condition should be maintained for **1** hour.

19 20 21 22 23 12. At the end of 1 hour, the reference standard is read as closely as possible to two decimals and recorded. Each meter in the test circuit is read to as closely as possible to two decimals and recorded **(AS** Found). The percentage of error is calculated by dividing the meter under test reading by the standard reading. Any meters under test that register above or below the reading of the reference standard should be adjusted to as close as possible to 100% accurate.

24 25 If it is necessary to adjust any meters in the thermal gang board, the test load must be maintained throughout the following procedures. Any adjustment to the full-scale should be 13.

1 done through the hole in the test cover located on the right side of the meter as one faces the $\overline{2}$ meter. This prevents cool air from rushing into the meter that would otherwise occur if the 3 canopy were removed for adjustment that would affect the temperature differences in the thermal elements. A flat-slot screwdriver is inserted in the full-scale adjustment screw. If the meter is to $\overline{4}$ 5 be adjusted upward on the scale, the screw is turned clockwise to the desired point. If adjusting 6 downscale, the adjustment screw is adjusted counterclockwise past the calibration point then *7* slowly back to the calibration point. The black maximum pointer should be in contact with the 8 red indicating pointer. This allows for any backlash, which could occur. If an adjustment has **9** been made, and it is desired to check accuracy of adjustment, reset the red and black needles 10 slightly down scale, this places the black needle back in contact with the red needle. The meter should be maintained at test voltage and current for an additional 45 minutes. At the end of 45- 11 12 minutes if the meter does not read accurately readjust the meter again and repeat the 45-minute 13 check again.

14 **What are the steps at which an error could occur?**

15 16 1. To begin with, the known accuracy of the board standard must be confirmed with a transfer standard from the National Institute of Standards and Technology (NIST).

17 18 19 20 2. The standard must be in the same circuit as the meters under test. This can be accomplished most conveniently by using an electronic auto ranging meter programmed to replicate the thermal response curve. Otherwise it is most likely a correction factor must be applied between the thermal board standard and the meters under test.

21 22 23 24 25 3. The zeroing of the meter is important to the accuracy of the full-scale test if the full-scale test is performed in the lower half of full-scale. A thermal demand meter's zero accuracy can influence the lower portion of the scale more so than the upper half of full scale because any deviation in accuracy at zero will decline as the meter is tested higher on the fullscale.

I 2 *3* 4 *5* 6 7 8 9 10 11 12 13 4. Maintaining proper test voltage and current is somewhat critical if the standard is of the thennal type. If the response curve of the standard is not exactly that of the meters under test, the standard could read above or below the meters under test. It should be noted that FPL's thermal board standards do not utilize the black maximum pointer. This can have two effects. First, without the black maximum in contact with the red instantaneous needle there is less resistance in movement of the red pointer that may result in a standard registration slightly higher than the indication that would occur if the black maximum indicator were in contact with the red pointer. Second, on the other hand if the voltage and current are not maintained closely and they are allowed to drift low over the test period it is possible the maximum point of the standard may not be the maximum point reached by the meters under test. That could result in the standard indicating a reading lower than obtained during the test period. That is why the preferred method of testing would be with an electronic auto-ranging meter of known accuracy. It would always read accurately to the maximum level of energy recorded over the test period.

14 1s 16 17 18 19 20 21 22 23 24 *5.* Reading the standard board meter and the meters under test can influence the relative reported accuracy of the test results. The thermal standard utilized by FPL has a resolution of 100 increments. Therefore if read to the nearest increment without interpolation the test result would be skewed one way or the other. To aid in making this point I have reviewed a 56 page report on test results of all -3,900 1V thermal demand meters completed in early 2003. In that report the standard reading was read at even increment in all 3,900 tests except for 49 tests, which read at $\frac{1}{2}$ increment readings. It is highly unlikely that the standard meters maximum indicating needle pointed to an exact increment in 99% of the tests. The same would be true for reading the meters under test. To yield an accurate assessment of the meters being tested, their maximum indicated reading must be interpolated as closely as possible. Otherwise their accuracy will be skewed one way or the other.

2s

1 2 3 4 *5* 6 7 6. It has been pointed out that some of the meter test technicians at FPL physically tap the thermal board standard meter at the end of the one-hour test period. The standard should be of a known accuracy and should not require any external manipulation to acquire an accurate reading. According to Mr. George Brown who has witness a number of tests at FPL's meter test center, that tapping of the reference standard has always resulted in the standard reading slightly higher. **A** higher standard reading skews the accuracy of the meters under test as well as the standard reference meter.

8 9 10 11 12 13 7. The utilization of a test cover is critical for accuracy and efficiency when a meter must be adjusted. However, if the cover is removed and cool air rushes into the meter the hot coil or element could be influenced greater than the cold element. If the hot element cools slightly and begins to drop slightly and at the same time a technician is attempting to adjust the meter upward or downward, he will be chasing a moving target. It would be impossible to adjust the meter accurately.

14 15 16 17 18 8. If the above were to occur and the meter is not allowed to continue at rated load for 45-minutes it is unlikely a miscalibration would be detected. The meter is designed to respond to 99.9% of any change over a 45-minute period. That is why it is recommended by Landis & Gyr to leave the meters under test at test load for an additional 45-minutes if adjustments are made.

19 20 **Did you review the written materials that FPL used to train its metermen regarding how to properly calibrate a thermal demand meter?**

21 22 Yes, I reviewed some sheets that were attached to the thermal meter test board. I also reviewed FPL test plans and procedures.

23 24 **Do you have any concerns about mistakes being made during testing and calibration of meters at FPL's meter testing center?**

25

 $\overline{1}$ Yes, I do, based on my review of some of the depositions and FPL documents. 1 have not $\overline{2}$ yct been granted access to the FPL test meter board or the individual meters, but hope to have the $\overline{3}$ opportunity to review them before or during the hearing.

 $\overline{4}$ **What concerned you?**

5 I was concerned about a number of things:

6 Brian Faircloth, who has tested many thermal demand meters at FPL, when asked about *7* the Landis & Gyr manual, which spells out recommended procedures for calibrating meters, testified that he had never seen the manual before. (Exhibit B, page 30, lines 22-25). He 8 9 testified in his deposition he does not follow FPL's procedure as posted on the meter board and 10 that lie taps the cover of the standard and has instructed others to do the same. (Exhibit B, page 11 48, line 8 through page 50, line 10.)

12 13 14 15 16 17 18 19 Furthermore, with Mr. Faircloth's testimony, he says every meter he tests goes out of his shop at 100% (Exhibit B, page 25 line 22 thru page 26 page line 14), that he calibrates every meter to 100% (Exhibit B, pages 53 and 71); however, test records provided by FPL to the PSC in response to questions posed by PSC staff and as supporting their allegation that 1V meters gradually go high and low over time, shows that a JC Penny meter number 1V-5879D last tested in 1999 by Mr. Faircloth, was tested as found at 2.28 and was left at 2.28. (FPL answer to staff request for data 8-18-2003, attached hereto as Exhibit D). I question whether Mr. Faircloth does calibrate EVERY meter.

20 21 22 23 24 25 I also noted that FPL did not use a test cover when calibrating thermal demand meters. The manufacturer indicated accuracy and efficiency is improved by using a meter test cover when calibrating a thermal demand meter. The meter test cover keeps the heat contained within the meter and allows for the meter to be adjusted carefully and precisely. Landis & Gyr states specifically: "The efficiency and accuracy of calibrating thermal demand meters can be improved by the use of test covers that have 3/8 diameter holes located over the zero and full

1 2 3 4 *5* 6 7 8 9 10 13 12 13 14 15 scale calibration adjusting screws, allowing the meter to be calibrated at zero and the calibration point without removing and replacing the cover.") Using a test cover improves accuracy when calibrating a meter for a couple of reasons. When the cover is removed from the meter, the cooler outside air rushes in and cools the so-called hot element of thernial unit much faster than it does the cold element. This causes a rapid change in the reading of the meter. FPL decided not to use this recommended test cover. Instead, it would have its testers remove the actual canopy cover, allowing the heat to escape from the meter itself, and then hurriedly make a full scale screw adjustment. FPL's test plan states "When necessary to make an adjustment, do so as quickly as possible and put the canopy back on the meter so as not to lose the heat." (maximum 20 seconds)." Not using test covers allows the cool air to affect the meter, and rushing to make an adjustment, time after time, is likely to lead to more mistakes than if a test cover were used. The accuracy of the meters was affected by the failure to use test covers. See Landis & Gyr manual (attached hereto as Exhibit E). I believe that it is somewhat telling, according to FPL documents, that 15% of its V class meters failed outside the range of tolerance. SEE 160 TDM (attached hereto as Exhibit F).

16 17 18 19 20 21 22 23 24 25 I was also concerned when I learned upon reviewing the deposition of Brian Faircloth, the FPL meterman who tested around 8,000 thermal meters. Mr. Faircloth testified that when adjusting calibration adjustment screws, he would bring the meter directly to the point of adjustment without compensating for backlash. (Exhibit B, pages 103-106.) The proper method, as clearly indicated in the Landis & Gyr manual, is to move the indicating pointer downscale past the calibration point and then adjust the indicating pointer up scale very slowly to the point of calibration with the maximum pointer in contact with the indicating pointer. This helps compensation for any backlash. (See Exhibit F.) This failure to follow the adjustment procedures outlined in the manual is, to me, further cause for concern that these meters were miscalibrated.

 $\mathbf{1}$ I noticed another instance in which the policy for calibration posted by FPL on its meter $\overline{2}$ board, which the metermen were supposed to follow, spelled out a key procedure in a much \mathfrak{Z} different way than recommended by the manufacture of the meter. Specifically, FPL's meter test $\overline{4}$ board procedure, step 10, states: "If a meter has been adjusted, the test board should be left 5 energized, with a stable load, for approximately 10 minutes, to check for proper calibration." 6 See Meter Test Center Operations, 9-23-93 (attached hereto as Exhibit G) and undated document $\overline{7}$ entitled Thermal Meter Board Procedures (attached hereto as Exhibit H). The Landis & Gyr 8 manual, at page 5 of the section related to Calibration of Thermal Demand Meters, indicates that 9 if the calibration point is going to be rechecked after the cover has been removed and replaced, 10 the present load on the meter must remain constant for a minimum of 45 minutes after replacing 11 the cover before a reading is taken. This indicates to me that FPL's calibration procedure in this 12 respect was not in keeping with the specifications of the manufacturer's manual for calibrating 13 thermal demand meters. Since FPL only waited "approximately 10 minutes" as compared to the 14 manufacturer's recommended "minimum of 45 minutes" the effects of the cool air on the meter 15 were likely to have more of an impact on the proper calibration of the meter than if FPL 16 metermen had followed the manufacturer's instructions and waited at least 45 minutes.

17 18 19 20 21 22 23 24 Given the failure to use a test cover, the need to quickly make adjustments and replace a canopy on a meter within 20 seconds, the failure to follow the procedures for calibrating a meter by waiting only 10 minutes, not 45-minutes when checking for proper calibration, the failure to set the calibration point by moving past the calibration point and then slowly adjusting upward to that point as recommended by the manufacturer, the fact that at least one key FPL meterman had never seen the Landis & Gyr manual, and thus not seen the calibration procedures contained in that manual, all add up to make it likely that the meters in this docket were miscalibrated and thus overregistered demand prior to the date of placing the disputed meters into service. This is 25

1 2 especially so when one considers that there is really nothing that can cause these thermal demand meters to over-register gradually over time.

3 4 *5* 6 7 8 9 One final note as to why I believe this case involves meter calibration error. In my experience around meter testing operations, if things are misplaced and not handled properly, it is often reflective of how a meter test shop is run and is likely to reflect a lack of attention to detail. I noted that FPL's internal document 0162-0164 TDM (attached hereto as Exhibit I) indicates that FPL lost or could not locate 60 1-V thermal demand meters that were supposed to be tested. These meters were lost after the entire class of I-V meters failed testing, so you would expect particular care would be paid to the status and location of these meters.

10 11 12 The factors set forth above, when viewed in a cumulative fashion, suggest that the evidence supports the thermal demand meters in this docket over-registering from the date of installation as compared to going bad gradually over time in the field through some unexplained 13 reason.

14 **Did anything else indicate to you that meters in dispute were miscalibrated?**

15 16 17 18 19 20 21 22 23 24 25 Well, as noted above, a lot of other things point in that direction. If you review the billing records of the accounts involved, once the thermal demand meter was replaced, all of the accounts experienced a significant decrease in demand compared to the demand levels registered previously. These thermal demand meters are all essentially the same. In one case, the Kings Point account, the customer retained his own billing records. Reviewing these records, and the graph that Mr. Brown prepared, permits one to view the energy demand before the thermal meter was installed, view the demand readings during the entire time a confirmed erroneous thermal demand meter was in use, and then see the significant drop in demand once the thermal demand meter was replaced. This indicates that the demand reading was high or overregistering for the entire time that the thermal meter was being used. Again, I don't believe that FPL will dispute

1 that this type of evidence suggests you can ascertain the point in time in which a change in

2 metering did occur. (See Kings Point billing, history and chart, attached hereto as Exhibit J).

3 **Why not?**

4 *5* 6 7 8 9 10 11 12 13 Well in reviewing certain of FPL's own internal documents, they appear to recognize that a customer's before and after demand readings are meaningful in determining the amount of refund that should be provided. For example, in FPL document 0161 TDM that starts with the phrase "1 V meter issues", the following question is asked: "What are the conditions that must be satisfied to provide a refund greater than 1 year?" After a reference to Rule 25-6.103(1), FPL states: "FPL methodology - Compared new electronic demand readings to similar months in the previous years to determine if error could be identified; if not, was there a material/consistent difference in the "new" and "old" demands? If so, offered refunds back over that period. Used higher of meter test results or "new vs. old" readings; used average difference for affected years;" (see Exhibit K attached hereto).

14 15 **Have you reviewed the billing records of the meters in dispute in this case, including comparing new electronic demand readings to similar months in the previous years?**

16 Yes, I have, for all customers.

17 **What has that reviewed indicated to you?**

18 19 20 21 22 23 It reflects that the demand meters were in error for a considerable period of time longer than 12 months and that the meters were likely misreading when installed. It also indicates that if FPL used this approach which I presume they did, that it probably should be applied to the meters in this case, since those meters reflect a difference that is both material and consistent in the new demand meters versus the old thermal demand meters. I would think FPL would want to treat similarly affected customers the same.

24

1 2 3 **thermal demand meters? There is an issue in this docket concerning what impact the sun can have on the thermal demand meters. Are you aware as to whether or not the sun can have an impact on**

4 *5* 6 7 8 9 10 11 12 13 14 15 16 17 18 19 The thermal demand meter is affected by heat, so yes, it is possible for the sun to have an impact. At Duncan Landis $\&$ Gyr, it was recommended that meters installed in states with extreme heat, such as Florida and Arizona, use sun shields to minimize the sun's impacts on thermal demand meters. I know that one particular meter, the Commercial Insulated Door account showed the effect that the sun can have on thermal demand meters. It should also be pointed out that FPL document 66-1 13 TDM "FACTS ABOUT DEMAND METERS" (attached hereto as Exhibit L) which is a scholarly article on thermal demand meters clearly reflects that the sun can have an impact on thermal demand meters. It states in document 96 TDM as follows: "A sun shield placed over the measuring element (Figure C-28) assures that direct rays of the sun will not produce an ambient temperature difference between the coils." Also, an email from an FPL employee, Jim DeMars states, "If potential is applied to the meter and there is no current flow, thermal meters have demonstrated the ability to register a little demand due to thermal heating from direct sunlight." FPL Doc. 158 TDM (attached hereto as Exhibit M). Thus, based on my experience, coupled with these recognitions that the sun can impact thermal demand meters, I have to say that the sun can cause the thermal demand meter to register a slightly higher demand than would otherwise be the case.

20 **Is this significant in your view?**

21 22 Well, if I was a customer who had a meter over-registering due the solar influence I could be over billed and a shop test would likely never detect there was any error on my meter.

23 **Do you have concerns about the accuracy of FPL's meter test boards?**

24 25 Yes, I do. I was involved in testing certain FPL meters in an independent test in Bradenton, Florida. These nine meters had previously been tested at the FPL Meter Test Center,

I and had tested high in the neighborhood **of +1.83%** to **+3.83%** with an average of **+2.7%.** (See **2 METERSFORINDEPENDENTTESTING3-29&302004REV.XLS** attached hereto as Exhibit N) 3 FPL brought the previously tested meters with them to the independent testing center. FPL also $\overline{4}$ brought with them a traveling standard that was tested against the standard in the independent 5 test board. The two standards matched. When the disputed meters were independently tested, the range of error on the meters tested in the neighborhood **Of-3.7%** to **+3.3%** with an average of 6 **7** +. **25%. If** the two meter test boards were both accurate, you would not see this type **of** disparity **8** when replicating a test. The meters were sealed following the independent testing, since **^I 9** understood the parties would return to Miami to test the meters again on FPL's test board to see **10** if the meters again tested high in the neighborhood of $+1.83\%$ to $+3.83\%$. If this were the test **11** result, it would suggest a problem with either the FPL test board or the independent test board. **If 12** the sealed meters were returned to Miami and tested on the FPL test board, and measured in the **13** neighborhood **of -3.7%** to **+3.3%,** consistent with the independent test board results, this could **14** mean the meters may have been tampered with from the point in time they were originally tested **15** in Miami to the point in time they were tested at the independent test board. After all, none of the **16** meters were sealed when they arrived in Bradenton. You will note, in my guideline for proper **17** calibration of a thermal meter, an inspection **of** the meter is conducted to detect if any tampering **18** may have occurred. I understand that FPL was not willing to retest these meters on its Miami test **19** board and allow the independent standard meter used in the Bradenton test to be compared to the **20** standard meter used at the Miami Testing Center thermal test board.

21 22 23 24 **25** The most telling information related to accuracy **of** the thermal standard meter is found in FPL Doc. **149-150** TDM. That document is a report **of** tests conducted on June **12,2002** on the meter removed from Commercial Insulated Door **of** Sarasota. FPL's Jim Teachman attempted to replicate the effect of heat from the sun on that meter to determine if heat could cause a thermal meter to over register. Three meters were involved in the tests: The thermal board standard, the

meter in dispute form Commercial Insulated Door and an electronic meter. According to the $\mathbf{1}$ $\overline{2}$ report four tests were run in sequence. Oddly enough, the thermal standard and the electronic 3 meter never matched. In fact their degree of difference ranged from 1.1% to 1.84%. I cannot conclude which meter was wrong. Perhaps if permitted to review the themial test board and $\overline{4}$ 5 standard that will be determined.

6 I also reviewed the deposition transcript of Mi-. Dave Brondey who was asked questions $\overline{7}$ about this testing sequence. He indicates that he is not willing to let the independent standard 8 meter be tested against the FPL standard meter at its thermal meter test board in Miami. When 9 asked if an investigation was conducted into the disparity between the test results in Bradenton 10 and the original test results in Miami, Mr. Bromley said he thought that information was 11 privileged and refused to answer any more questions on the subject. See Deposition of David 12 Bromley, page 68-74 (attached hereto as Exhibit 0).

13 14 15 16 Finally, in reviewing the deposition transcript of Mr. Faircloth, who had worked in the meter test center for over 6 years, since March of 1998, tested around 8,000 thermal demand meters, and presumably would be aware of events affecting the thermal test board meters, I was surprised to read the following at page 95 of his deposition (see Exhibit B):

17 Q. Do you know when the - How often the standard meters are tested or checked?

18 **A.** No.

19 Q. Have you ever tested a standard meter for accuracy?

20 **A.** No.

21 Q. Do you know if anybody who has tested a standard meter for accuracy?

22 **A.** No.

23 24 25 So, given what I have described, I have concerns about the accuracy of the meter test board. I understand that there may be some efforts to review those meter boards, and if allowed

 developed at hearing. **Does this conclude your testimony?** 4 Yes. to participate in those reviews, assuming they are permitted, In my opinion may be further

la Para de Caraca de Santo de Caraca de Caraca.
Nacimiento de Alexandro de Caraca de Caraca de Caraca.

Worldwide Reporting Service
Miami & Orlando & Tampa

2 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 87 right, they do have loose bearings, or could you say, I don't know? A. I might be able to look at it and tell. (2. What would you look to? A. Now, as far as running fast, the load I'm talking about running fast doesn't have to do with the demand, it has to do with the KWH. That comes to where the disc is in here (indicating). It's got a shaft that goes straight through, and it sits on what they call jewels, and that's some people call it a bearing, some people, actually a jewel. It's a little needle. And the disc floats in between, sits on that. Q. That only applies to KWH? A. That's for your KWH. Q. How about KWD, anything that could mechanically cause it to run fast that- you're aware of? A. Not that I know of. Other than adjustments. Q. Do you know why, if FPL *is* continuing to use thermal demand meters? A. I know there's some still in the field. That's all I know. Q. Do you know if any of them are being put back in the field? When you test them are any of them going back out?

Worldwide Reporting Service
Miami & Orlando & Tampa

 $\label{eq:2} \frac{1}{\sqrt{2}}\sum_{i=1}^N\frac{1}{\sqrt{2\pi}}\left(\frac{1}{\sqrt{2\pi}}\right)^2\frac{1}{\sqrt{2\pi}}\left(\frac{1}{\sqrt{2\pi}}\right)^2.$

5

8

9

25 2 at atime? 3 **A.** No. 4 Q. You say you had to fine tune them when you 6 that adjustment to zero? 7 **A.** Be able to see. That's it. 10 11 12 amount in the wall? 13 **A.** There's no tightening. It goes one way or 14 the other. 15 16 17 18 Q. Right. 19 **A.** On this one it says what, full scale 20 (indicating). 21 Q. Right? 22 **A.** So this is supposed to run at 2, and I've 23 adjusted this to zero with the zero screw on the back, 24 right, this is supposed to read 2, according to all my 25 calculations on the computer, tells me it should be Q. Have you ever seen anybody not do them one make this setting. Is there skill involved in making Q. Again, I don't know what these are like, but would it be like if I were screwing in a screw on a piece of wood, you just tighten it down all the way, or do you try to tighten it so the screw is a certain This arm right here, if this is sitting, you take this screw (indicating), this is your zero screw. Do you see where it says zero?

1 reading 2 at what I'm testing it at, this screw, and 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 it's coming up here and it's 1.95, right, this screw here will adjust it to 2. All you do is go tweak (indicating). *You* go through and do all of them, you let them sit. What you're doing is you're making sure when you come back, if it's still sitting on 2, you don't touch it. Maybe for some reason a little height, whatever, I don't know. Maybe it went over a little bit. So what you want to do is fine tune it back to 2. Because you want that meter to go out at 100 percent. That's what I did with every meter I ever tested (indicating). Q. The part where you were describing about 2 and being close to 2, what not, that's the calibration part? **A.** Right. Q. And in part, taking the red number to zero, 20 that's the -- 21 **A.** Pre-heat. That's the zero adjustment. 22 That's why it has zero below it, so you know 23 (indicating). 24 Q. Does it matter when you pre-heat those

25 meters, you can pre-heat them and then you take them

2 Q. But you don't know whether the needle

3 adjusts slightly.

- 4 You said it moves light slightly. You don't
- *5* know whether it's a slight movement up or downward?
- 6 **A.** Didn't pay attention.
- 7 Q. I presume if we were to go into that lab and
- 8 test and tap, we could probably see which way it would
- 9 move, couldn't we?
- 10 **A.** I assume.
- 11 Q. Do you know what the tapping of the meter
- 12 does?
- 13 **A.** No.
- 14 Q. Have you ever tapped the meter being tested?
- 15 **A.** No.
- 16 Q. You only tapped the standard?
- 17 **A.** Correct.
- 18 Q. Do you know if that's a practice that's
- 19 commonly used in the meter testing center?
- 20 A. I don't know.
- 21 Q. Do you know if it's in the policies and
- 22 procedures?
- 23 **A.** It's not there.
- 24 Q. Have you ever given instruction to anybody
- 25 else as to how to test thermal demand meters?
- 2 Q. As part of the basic instructions have you
- 3 told them to that he should tap the standard meter?
- 4 A. Correct.
- 5 Q. And who have you told that to?
- 6 **A.** I believe Mister -- I'll say I don't know,

7 because I don't remember who.

- 8 Q. You don't remember who, but you remember --
- 9 A. I don't remember if I said it. In other
- 10 words, I don't know. I don't remember that exact day.
- 11 Q. But if we were trying to figure out the best
- 12 way to test meters, and the Commission, the Public
- 13 Service Commission asked you to make sure you have an
- 14 accurate test, should you tap the standard meter, what
- 15 would your answer be?

16 A. I would tap it.

- 17 Q. And the reason is?
- 18 **A.** It's the way I was shown.
- 19 Q. Do you know, has the protocol for testing V
- 20 meters changed recently?
- 21 **A.** What do you mean?
- 22 Q. Has there been any change with respect to
- 23 the V meters when you test them, what you do with them
- 24 after you test them?
- 25 **A.** I test them, and I put them on a rack and I

- 19 these V meters gradually or suddenly read high --
- 20 **A.** No.

9 have?

 10

14

15 **A.** No.

16

17 **A.** No.

- 21 Q. $-$ In the field?
- 22 **A.** No.
- 23 Q. If a meter was miscalibrated and then put
- 24 out to the field, you would agree that could result in
- 25 erroneous readings.
- 1 documents we've identified?
- 2 **A.** You mean would I test it differently from
- 3 the way the procedure tells me to test it?
- 4 Q. Right.
- *5* **A.** Besides the fact that I adjust all my meters
- 6 back to 100 percent? No.
- 7 I follow the procedures. The only thing I
- 8 do is I like 100 percent meter to go out to the
- 9 customer.
- 10 Q. Do you know why there was a comment in the
- 11 document I showed you about not adjusting it if it was
- 12 within plus or minus 2 percent?
- 13 A. I do not know.
- 14 Q. Have you ever seen the meter test plan,
- 15 FPL's meter test plan?
- 16 **A.** No.
- 17 Q. The thermal demand meters, the V meters, how
- 18 were they tested in the old days with respect to the
- 19 meters going on the front of the board?
	- Would you intermix the high scale meters
- 21 with the low scale meters?
- 22 **A.** How did I test them as to calibrating?
- 23 Q. Yes.
- 24 A. Did I put high scale and low scale together,
- 25 yes.

25 Q. So you don't follow it?

BEFORE THE FLORIDA PUBLIC SERVICE COMMISSION

DOCKET NO.: 030623 FILED: April 28, 2004

IN RE: Complaints by SOUTHEASTERN UTILITIES SERVICES, INC., on behalf of various customers against FLORIDA POWER and LIGHT COMPANY concerning thermal demand meter error.

....................... X

9250 West Flagler Street Room 1606 Miami, Florida 33174 May 5, 2004 $8:50$ a.m. $-12:45$ p.m.

DEPOSITION OF JIM TEACHMAN Taken before Michael Jay Kugler Notary Public in and for the State of Florida at Large, pursuant to Notice of Taking Deposition in the above cause.

 $1 - \epsilon$

r
Teachman-Complainant-Direct 95 1 Q I've been asking you. You've been in the meter shop for twenty 2 3 years. In your experience -- $\overline{4}$ A I've seen it both ways. 5 *6 Q* t been more high han low, or more low 7 than high? 8 A **I** don't keep a record of that. So you don't have any idea whether there 9 \overline{O} have been more high readings than low readings? 10 MR. HOFFMAN: Objection, asked and 11 12 answered. No, **I** don't. 13 A An error as a percentage of full scale due 14 Q to a full scale adjustment condition, will it increase 15 as the meter's tested closer to full scale, based on 16 your experience? 17 MR. HOFFMAN: Object to the form of the 18 question. 19 20 Vague and ambiguous. 21 A Unfortunately you're killing me with your *22* quest ion. 23 What was **it** again? *Q* An error as a percent of full scale due to 24 a full scale adjustment condition, will it increase as 25 Worldwide Reporting Service

Miami ∞ Orlando ∞ Tampa

Teachman-Complainant-Direct 96 the meter is tested closer to full scale? 1 $\overline{2}$ **MR.** HOFFMAN: Reiterate the objection, and it calls for speculation. 3 A It actually sounds to me like you're asking $\overline{4}$ the same question you've already asked me. 5 6 If the error is four, it stays four. 7 That's the way I understand the question you just read to me. 8 9 Q You consider yourself an expert in meters and meter testing? 10 No, I don't. 11 A 12 I wouldn't be working for FP&L if I was. 13 I'm sorry, that slipped. Do you know what could cause the thermal 14 \overline{O} element of a thermal demand meter to gradually read 15 high, if anything? 16 A No, **I** don't. 17 Q Do you know or are you aware of anything 18 that could cause the thermal element of a thermal 19 demand meter to suddenly read high? 20 A No, I don't. 21 Q **If I** asked you the same two questions with 22 respect to a thermal demand meter, either gradually 23 reading low, or suddenly reading low, would your 24 answers be the same? 25

Worldwide Reporting Service $Miami \n\in \text{Orlando} \n\in \text{Trabola}$

Worldwide Reporting Service
Miami ∞ Orlando ∞ Tampa

-

1 2 *3* 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 Teachman-Complainant-Direct 98 Q HOW much did it cause the meter to misread, if you will? A It varied. Q Within what ranges? A Because I didn't look at it, at the percentages. It could go from one, I don't know if the word is tick or line, to four lines. Q Do you know if the meters that you tested in the lab, were they ever field tested? A Not to my knowledge. Q Who would know if they were field tested? A I have no idea. Q Would you know if they were field tested, typically? A Only when George Brown requested the ones, but not the ones that **I** tested. Q Do you know if FP&L ever had a field test unit on the trucks, that they could use in the field? A For what? Q To test the accuracy of these thermal demand meters? A Not to my knowledge. Q We talked about the sunlight issue. Do you know what could cause a thermal

Worldwide Reporting Service
Miami & Orlando & Tampa

Worldwide Reporting Service
Miami & Orlando & Tampa

RUTLEDGE, ECENIA, PURNELL & HOFFMAN

PROFESSIONAL ASSOCIATION ATTORNEYS **AND** COUNSELORS AT LAW

STEPHEN *A.* **ECENIA RICHARD M. ELLIS** KENNETH **A. HOFFMAN THOMAS W. KONRAD MICHAEL** *G.* **MAIDA MARTIN P. McDONNELL J. STEPHEN MENTON**

POST OFFICE **BOX** 551, 32302-0551 **215** SOUTH MONROE **STREET, SUITE 420 TALLAHASSEE, FLORIDA 32301-1841**

> **TELEPHONE (850) 681-6788** TELECOPlER *(850)* 681-6515

August 18,2003

R. **DAVID PRESCOTI HAROLD** F. **X** PURNELL **MARSHA€ RULE GARY R. RUTLEDGE** R. DAVID PRESCOTT
HAROLD F. X. PURNELL
MARSHA E. RULE
GARY R. RUTLEDGE
GOVERNMENTAL CONSULTANTS **MARGARET A. MENDUNI M. IAN€ STEPHENS**

HAND **DELn'ERY**

Cochran Keating, Esq. Division of Legal Services Florida Public Service Commission 2540 Shumard *Oak* Boulevard Room 370 Tallahassee, FL 32399-0850

Re: Docket No. 030623-E1

Dear Cochran:

Enclosed is **an** origmal **and** one copy of Florida Power & Light Company's Responses to the Commission Staff Data Requests dated July 29,2003. Enclosed also is a copy of FPL's Notice of Intent to Request Confidential Classification of FPL's response to Staff Data Request No. 3. The unredacted response to Staff Data Request No. 3 has been filed with the Commission Clerk in **an** envelope marked "CONFIDENTLAL."

Sincerely,

Kenneth A. Hoffman

KAWrl

 $cc:$ \blacktriangle Daniel Joy, Esq., with enclosures Robert Vandiver, Esq., with enclosures Mr. Bill Feaster, Esq., with enclosures

FPL\keating.818ltr

FPL's RESPONSE TO COMMISSION STAFF DATA REQUEST RECEIVED JULY 29,2003 DOCKET NO. 030623-E1

1. Please provide all data or analyses, from the meter manufacturer(s) or other sources, that would support the conclusion that the only way thermal demand meters **in** question may over-register (i.e., read too high) is if the meter was improperly calibrated at initial installation. Alternatively, provide all data or analyses to support the conclusion that such meters may gradually or suddenly read too high over time, even if properly calibrated at the time the meter was set. Include any engineering analyses, articles from journals, trade publications, or expert testimony, including documented experience of other utilities.

> FPL does not have or know of any data that supports the conclusion that thermal demand meters only over-register if the meter was improperly calibrated at initial installation.

> To support that meters may gradually or suddenly read high, FPL offers the following information: (1) **A** presentation by **A.** R. Jenny of Westinghouse Electric *Corp.,* Meter Division, used at a Public Utility Short Course for Electrical Metermen at the Oklahoma State Meter School, Oklahoma State University; **and** (2) FPL's own experience and test results.

> (1) Attached in the presentation mentioned above, the author discusses the various components of a thermal meter. In particular, two bi-metal springs, which act as a thermometer, are the "heart" of the thermal meter. The difference in the heat of these two opposing bi-metal springs is proportional to the power of the load being measured and it is this difference that is measured on the scale. On page 28 of the presentation, referring to these two bi-metal springs, it is noted,

"The heaters themselves are closely matched and have nearly the same resistance when installed in the circuit. Any differences in their resistance may be compensated for by the adjustments of the meter".

FPL suggests that the author is acknowledging that calibration adjustments may be necessary. FPL also notes that the author does not state that these adjustments will only go in one direction.

 $\mathcal{L}_{\mathcal{F}}$ and $\mathcal{L}_{\mathcal{F}}$

(2) FPL's own experience indicates that thermal meters can gradually read high or low after being calibrated. FPL believes that is the reason that adjustment screws (screws that can be turned in either direction) were placed on these meters. This is unlike a solid state meter, which has no such adjustment mechanism for **the** user. Below, FPL has incIuded meter test results for some of **Mr.** Brown's clients. In Columns D and J (Test - **As** Found % Error) you will find the % error when the meters were brought in from the field and tested. In Columns E and K (Test - As Left), it shows the meter test results after the meter was calibrated. **As** can be seen, multiple tests for the same meter over time indicate that these meters can drift in either direction once they have been calibrated.

 $1 - 1 \begin{pmatrix} 1 & 1 \\ 1 & 1 \\ 1 & 1 \end{pmatrix}$ PRINCIPLE OF THERMAL AND MECHANICAL DEMANDS By A. R. Jenny Westinghouse Electric Corporation ing a Meter Division Raleigh, North Carolina \mathcal{P}_S 11 \mathbf{I} For Presentation at \mathbf{v} Public Utility Short Course \sim $\!$ for $\alpha = 1$. a de la Serballe.
Com Belleman Electrical Metermen Oklahoma State Meter School $\mathbf{V}^{(1)}$ Oklahoma State University March 24, 1966 $\frac{1}{2}$, $\frac{1}{2}$, $\frac{1}{2}$ \mathcal{C} $\sim 10^{11}$ km $^{-1}$

Thermal Demand Meters

 $-21-$

Introduction

As stated at the outset, there are three classes of demand meters which are Class 1, curve drawing instruments; Class 2, integrating demand meters or block interval, mechanical demand meters; and Class 3, lagged demand or thermal demand meters.

First of all, just what is a thermal demand meter? It has been described as a measuring device which transforms electrical energy to heat and measures this heat by a thermometer. To be more specific, the circuit is such that the difference in heat developed in two sets of heaters is proportional to the power of the load being measured. The thermometer (a pair of opposing bi-metal springs) measures the difference in temperature and indicates this difference on the scale. The scale, of course, is not calibrated in degrees but in kilowatts or numbers that can be readily converted to kilowatts by a simple multiplier. At this point, let's take a simple thermal meter, from the start describe all parts involved and see how each part contributes to the heat principle of determination of KW demand.

Theory

To start with, let's take a heater of a known resistance and attach it to a source of voltage of a known value. $If \cdot$ the voltage and resistance are so chosen, a current of sufficient value will flow through the resistance and will give off heat which is proportional to the current flowing through the resistance. Another way of saying this is, the amount of heat or watts dissipated would be proportional to the current and resistance or in formular form expressed as **的解放**中的新闻 T^2R . (See Figure 12.)

If we now take two pieces of metal with two different coefficients of linear expansion, join them together, and wind them in a helical coil, we form what is commonly referred to as a bi-metal coil or spring. Referring to Figure 13, if we place this bi-metal swing so that one end is anchored to a fixed object, attach a free moving shaft to the other end of the bi-metal with a pointer attached, and place this whole assembly in close proximity to the heater described in Figure 12 we would note that the bi-metal would expand, moving our free-moving shaft and pointer assembly as it expanded. Further, if we placed a scale behind our pointer, we could by careful calibration and control of the resistance and voltage determine the units of heat and thereby calibrate the scale and have in effect a thermometer. We can vary the degrees of deflection of the pointor by either raising or lowering the voltage or raising or lowering the resistance value of the heater circuit. In practice in meters, however, the voltage is for all practical purposes constant as well as

 $-22-$

the resistance of the heater circuit being constant with the variations in deflection caused by the load which is attached to the voltage supply and in whose circuit the resistance "R" is placed.

One of the difficulties of this type of circuit, though, is that ambient temperature could also affect our singular bi-matal as shown in Figure 13 and could cause some small degree of error. To compensate for the ambient temperature error, we wind a second bi-metal, placing it in direct opposition to the original bi-metal, as shown in Figure 14, so that any change in ambient temperature will drive each bi-metal with the same strength. Since they are in direct opposition, this would cancel any effect of the ambient temperature change.

As a matter of interest, Figure 14 is the actual basic component for a thermal ampare meter whose scale can be calibrated in ampares so as to deflact proportional to the current flowing through the heater circuit.

In order to produce a watt demand circuit so that the current flowing will produce heat to drive a pointer proportional to KW of the circuit, we need a somewhat different or modified setup which is shown in Figure 15.

Figure 15

So as to not confuse ourselves with the many parts and connections within the circuitry, let's take a look at the basic components and see what we have gained over those shown in Figure 14. First of all, we still have two bi-metals wound in opposite directions anchored to a fixed point in each case and attached to a movable shaft to which a pointer and scale assembly is set. The first change we note is that we now have two heater circuits whereas in Figure 14 we had a singular heater circuit. The next major change we see is that our voltage now comes from the secondary of a potential transformer whose primary is attached to a line voltage of some value. Looking at the current which flows in the loop formed by the secondary of the potential transformer and the two resistance heater circuits, we find that the value of this current could be found by dividing the voltage of the

 $-24-$

secondary of the potential transformer by the sum of the resistance in this circuit, or expressed another way, $I = E$. Note also that the current as shown in Figure 15 flows from left to right in both of the resistances. Now let's add a few more connections to our circuit as shown in Figure 16.

Figure 16

Notice that we have found a midpoint of the secondary of our potential transformer, brought out this point and attached it to the line side of our supply voltage. Further, we have taken the center point of our two resistances and connected it to one side of our load. The other side of our load we have taken back to the remaining line of our supply voltage. As our load is applied to the system, a current I_I will flow coming into the center point of our transformer's secondary and dividing it into two portions, IL both left and right.

The I_I flows through its respective resistance arm, rejoins \mathbf{z}

 $-25-$

at the midpoint of the heater circuit, and through the load *goes* **back** to the other side of the line. **Formerly,** we only had E in each of the two **heater** circuits;' **and as** explained, $2\overline{R}$

these two currents (since they are equal in magnitude and work on heaters **which** produce heat in equal proportions and they in turn act on bi-metals wound in opposite directions) produce no reflection of **scale.** in the **left** heater, **we.** note that the **arrows point** in the same Whcn the line **current** in **introduced** direction; therefore, **these** two currents will add. However, the currents in the right heater, as shown in Figure 16, are **in.** opposite directions and, therefore, will subtract **frcm** each other thereby producing a **lesser** cwrent than originally *was* found without the line current applied. Likewise, we would fiad that **the** lefthand portion would have a greater current because of the adding effect as noted before. Since **we** .are interested 5.n heat developed **in'each** of *the* heater . circuits, by mathematical manipulation **we can** see that **the heat in** the left **and** the **heat** in the right **heaters will add** :.. ~ . . ,. .. .- .. **as** seen in **the following** equations.

.. ..

u.

 $W_{L} = \left(\frac{E}{2R} + I_{L}\right)^{2}R = \frac{E^{2}}{4R} + \frac{E I}{2} + \frac{I^{2}R}{4}$

..

$$
W_R = E \frac{I_L}{2R} \frac{I_L}{2} = \frac{E^2}{4R} - \frac{EI}{2} + \frac{I^2R}{4}
$$

By adding **the** heat **in** the **left and the heat in the** right, we come out with the following equation.

-26-

$$
W_{\text{L}} - W_{\text{R}} = \left(\frac{E^{2}}{4R} + \frac{E^{2}}{2} + \frac{I^{2}R}{4}\right) - \left(\frac{E^{2}}{4R} - \frac{E^{2}}{2} + \frac{I^{2}R}{4}\right)
$$

$$
W_{\text{L}} - W_{\text{R}} = \frac{E^{2}}{2} + \frac{E^{2}}{2}
$$

$$
W_{\text{L}} - W_{\text{R}} = E^{2}
$$

Ths difference between the two comes out **E1** which **we** immediately recognize as being watts of the circuit.

 $-27-$

We might make note here that in the above equations when . **.we multiplied** the **current** and resistance together that we **should use** vector notation **so** that in actual practice when E and I are multiplied **the phase** angle **between** E and I **should be** taken into consideration where the **phase** angle **5s** actually .. **the** power factor of load.

We can by various examples both at unity and other than unity power factor show that the above condition applies.

.. . . -. At this time, we should make some mention of the components which make up the thermal meter as to their , . physical makeup as opposed to their location in the circuit. First of all, the potential transformer that we have been describing **is** actually a winding placed on the potential coil of the meter which in effect first reduces the size and number of components used and secondly cuts down the watts **loss** of the.potentia1 circuit in **general.** -. **As** described before, this potential circuit and its resulting **secondary voltage** is used to produce the **short** circuit current which flows at all times in the resistance circuit compoaed of **the two** heaters. **We** might **add** that this current **flows not only'at various loads but at a no-load condition.** .

.
. . . .

The value of the secondary voltage applied to the heater circuit is chosen to reduce the watts loss in the: circuit at **high** load condition3 on **the** inzter and to not overheat the resistances **which give ua** our up3cale **deflection** at **various loads** ,

The bi-netal **springs,** so **far** taken for granted, arc . ' really the heart of the **thermal** and must be closely natched both **os** to **torque** and deflection *for* **a given temperature change.** These **metals** are **usually** chosen with great care . with two bi-metal springs **wound from** the **same singular** piece. This is to achieve a near perfect match in deflection per degree temperature rise and to eliminate any possible error due *to* not *over* the ambient temperature but **the** heat **as** applied fron the heater circuits. piece. This is to achieve a near perfect match in deflect per degree temperature rise and to eliminate any possible error due to not over the ambient temperature by the heat as applied from the heater circuits. The heaters

hearly the same resistance when installed in the circuit. Any differences in **their** resistance3 may be compensated for by the adjustments of the meter,

So far, no mention has been made of the enclosure in which the **bi-metal springs** themselves **and** the heaters *are* placed,. The enclosure **must alao** be chosen with **care** insofar **as** material **and design** considerations **are concerned.** Ths enclosure *mus't* have a **czrtain** heat-storage **value so** *as* to **follow** the-specifications **cis** *were* **laid do%,** by MS-5 or **other similar** specifications. **In** general, the **sp2cification state3 (and** as stated in the *earlier-part* of **tho** paper) that **ths thermal** meter **should respond to load at a** rate **such that** *905*

.. **-28-** _.

a similar curve can be expected from a t30 meter.

It is also interesting to note that when the load is disconnected or line current goes to zero, the thermal indicator will go back down scale at a rate which closely approximates the rate of rise as when load was applied. That is, when a load is disconnected at the end of 15 minutes (on a tl5 meter), the pointer would have gone downscale approximately 90% of the value of a load applied at the time of disconnection.

Mechanical and Thermal Comparison

So far, we have concerned ourselves only with the single phase thermal demand meter. Thermal meters are made for a wide variety of applications--both single phase and polyphase. Thermal meters of course are available either as a separate unit or as a combination meter with the latter being the most popular of the types available. Thermal meters also are available as recording meters even though we have mentioned before only the indicating type. $\mathcal{L}^{\text{max}}_{\text{max}}$

Having covered the fundamentals of thermal meters and the mention of their various types, we are now in a position to evaluate their merits relative to other forms of meters such as the mechanical meter.

First of all, their costs, speaking strictly from the combination types thermal as compared with the mechanical type demand meter which is actually a demand attachment of the mechanical type, are the same.

Insofar as maintenance is concerned, the single phase thermal takes a lead in that it has been shown that maintenance of the thermal meter is remarkably low. Usually it is considered that the testing period can be the same as that of the watthour meter. However, the fundamental, of course, is that the only moving parts are the bi-metal springs, shaft, and the maximum demand pointers. It is almost axiomatic that the maintenance of demand devices varies directly with the number of moving parts. While this is only one of the factors in the choice of either

 $-30-$

a'thernal or mechanical, it **is generally** considered *30* **izportant** that it will outweigh other disadvantages. Actually, in a **thermal** meter there **are** no high **speed** shaft3, clutch **mechanisms,** motors, or'cam **arrangements.** In short , the parts *0s* : mechanical devices which require the bulk of maintenance are not present in the thermal.

Insofar as useful life **is** concerned, it **is** sometimes **closzly** allied **with** maintenance. **Here again,** the **single** phase **thermal meter** has a slight edge over the **mechanic& Ye** are still getting experience from the thermal meters since they were introduced,but **we** seem to lean a **little toward** the **thermal** in that a longer **life** is **obtainable by a thermal device. The** simplicity **of** the thermal **meter** again **iB its** main **asset as** the parts which wear out do not exist.

Insofar *cs* accuracy is **concerned, the** mechanical **is some**what in the lead as there are more factors involved in ascertaining accuracy on the **mechanical versus** thermal. That is. to say, **in** the **thermal meter, even** the **'selection of** the **bi**metal springs, **heaters, ad** the **housings are such'** that **we assume equality;** however, there is no such thing **as** a **100%** identical unit in their characteristics. This makes differences in **scale** distributions and **obvious inaccuracies. However,** this **is** not intended to imply that the thermal **meter is inaccurate** . but **merely** that it hmta **precision device aa in the** *case* **of** the mechanical **meter, Actually, this is one of** the cases of **getting something for** nothing, that **is, or jxall** . sacrifice *.in **accuracy** enables **us** to gain largely **in** other **respects.**

Insofar as versatility is concerned, the thermal can ' **be** adapted to **all types** of metcrinz; but on **the** *othcr* hand, the therrcal demand unit cannot be **pulled** off *and* **placed** on another type of neter as in the case of a mechanical demand **register.** This practice is an-important **factor** in **the** delcction of demand cquipaent with **some** utilities,

The **possibility** of **peak** aplitting by **block interval** device is well known and naed not be elaborated. that **a thermal mater** will **not split** peaks hac. often been decided **as a** major **advantage** of **thermal** actcrs. this reason **i3** certainly not **a** compelling one to **influence the use** of thermal meters rather than block interval **devices. Despite** all the argwent **thet** has been,advanced on **this score, the** fact, remains that actual conparison of the readings **of** two types of devices on the same load shown little difference in **most** installations. **This** is not **intended** to *discredit* the principle of the **thermal** meter but **to place it in** *it3'* proper. perspective. The fact However, .,

.

Insofar **a3** the **response** of the thermal **meter** is **concornod,** sometimes called logarithmic or expotential, the thermal meter **is** nzarly a duplicate of tho heating characteristics of **the** apparatus which must be provided to supply the load, that is, the transformers, generators, etc. Since the tenperature **rises, this equipment is also osssntially** oxpotential; **wa might** .. make this statenent, and while this is trua, the time interval for such equipment is in general a great deal longer than that of **a** mzter. **Also** the concordance of ths differcnt **types** of

-32-

domand meters illustrates that this factor is inatterial. If the two devices indicate the same maximum demand, the rate of rise is of no consequence.

The fundamental reason for the concordance of different types of devices on various loads is probably that so many intervals occur during a month that peak splitting is not a factor. While the general picture indicates that the thermal meter has a minor advantage in this respect, it is of course true that on certain types of loads it may have a pronounced advantage. Overall, however, these considerations are of minor importance.

Insofar as testing is concerned, fundamentally, a thermal meter of a given interval takes a longer time to test than a block interval device of the same interval. Furthermore. mechanical devices can be driven by a synchronous motor with no question as to the proper indications. Also, block interval devices can be conveniently tested in the field.

We might also add that in testing the increase of time does not mean that more man hours are required: since constant attendance is unnecessary. However, it does mean increased test facilities to some extent. Further, the thermal equivalent of a synchronous motor drive is an electric load controller which holds the load constant regardless of voltage fluctuations. The other alternate, of course, is the use of a standard with the same characteristics as the meter being tested. This is analogous to the portable standard method of checking watthour meters. Any load fluctuation affects the standards in test meters equally.

 $-33-$

. . Pågli testing in gemeral sessa to be gåving äver te lab testing as a more effective and efficient method and this probles does not seem to be a disturbing influence. However, there is certainly ream for argument among the various utilities about this particular point.

In conclusion, the above points tend to bring out some of the faatures of both the machanical and thermal metors and how they rate in general with various utilities. Further, in utility practices today, we can see the popularity of various types of both machanical and thermal meters based on the NETA figures. These figures show that the thermal meter is quite popular on single phase application and that the mechanical meter is more popular for polyphase application. The reasons for their popularity, of course, is not the purpose of this paper as it is strictly a mattor of choice of the utility involved and its experiences. Certainly, there is rocm for argument in either case with each side having their substantiated reasons for their choice.

 $-35-$

3. For Florida Power & Light Company **(FPL),** please provide detailed calculations used by **FTL** to determine the refund that it thinks is appropriate for each meter for which a refund is sought by SUSI under a pending complaint before the Commission. If billing history was used in any of the calculations, please show how such records were used. If formula was used to estimate months where no billing data was available with the new meter, please explain the method and all assumptions used.

Process steps used in developing response:

1. The meter test results were compared (if more than one test done) and the most favorable result for the customer was wed to calculate a credit or debit.

2. The result from 1. above was applied to the historical demand/usage for the 12 months prior to the meter change in order to calculate a rebill demand/ usage.

3. The result from 2. above was entered into FPL's billing system (transaction BCON) to generate a credit or debit.

4. The demandusage on the account after the meter was changed was compared with the historical demand/usage from the prior years, same months.

5. Within the scheme of negotiations, if the demand/usage level showed consistent and significant change $(>10\%)$ consideration was given to providing a credivdebit beyond 12 months.

6. The calculated % changes in demandusage were applied to the historical demand/usage for the period back to the meter change in order to calculate a rebill demand/usage.

7. The result from *6.* above was entered into FpL's billing system (transaction BCOM) to generate a credit or debit.

8. If the demand/usage level did not show a consistent and significant change $(>10\%)$, the calculated % change in demand/usage was applied to the historical demand/usage for the 12 months prior to the meter change in order to calculate a rebill demand/usage.

9. The result from 8. above was entered into FPL's billing system (transaction BCOM) to generate a credit or debit.

* Any differences in KW are due to rounding and/or BCOM (FPL Billing System)

Target

Account Name: Corporation

Tariff Rule Worksheet

 \mathbf{I}

 \mathbf{I}

Target

Account Name: Corporation

Tariff Rule Worksheet

Target

Account Name: Corporation

Tariff Rule Worksheet

Target Account Name:

Corporation

02873-11708

39242-15316

Account Target

Name: Corporation

02873-11706

39242-15316

4. For FPL, please identify and describe every known situation which **FPL** determined that **a** refund for a period greater than **12** months was appropriate, regardless of whether the situation relates to a pending SUSI complaint. What specific characteristics or events justified departure from the 12-month limit in Rule **25-6.103(1),** Florida Administrative Code?

> "Cause Codes" that identify the reasons for refunds are only retained on customers' records for a 24-month period. FPL was able to identify **137** customer accounts that had received **a** refund for a period greater **than 12** months, during the previous 24-month period. However, none of the **137** accounts reviewed, that were issued refunds beyond a year, were associated with "fast meters" **as** discussed **in** Rule **25-6.103(1), F.A.C.**

,.

 $\overline{\Omega}$

5. For FPL, please provide the detailed calculations that FPL would use to determine the appropriate refund for each meter for which a refund is sought **by SUSI** under a pending complaint before the Commission, based on strict application of Commission rules. Please identify all assumptions.

Process steps used in developing response:

1. The meter test results were applied to the historical demandwage for the **12** months prior to the meter change in order to calculate a rebill demand/usage.

2. The result from 1. above was entered into FPL's billing system (transaction BCOM) to generate a credit or debit.

		OOCKET OOOOCO – COMMISSION OTALT KEGAEST QUESTION O.			
				KWD or kWh	credited or
Name	Account	Address	First test results	credit	debited
		03 US Highway 301 Blvd STE			
J. C. Penney Company Inc.	07064-37886	Ό1	4.31%	206	\$1,829.89
J. C. Penney Company Inc.	90964-37216	!076 9th St N #Penneys	3.01%	о	\$0.00
Dilliard Department Stores	28011-72467	15 _ _ _ _ .			
Dilliard Department Stores	51180-46985	1441 Tamiami Trl # Dilliards	2.08 kWh	49266 kWh	\$2,262.61
Best Buy Co.	97114-18237	12395 W Sunrise Blvd	2.96%	٥	\$0.00
Best Buy Co.	30591-38093	20540 State Road 7	4.96%	292	\$2,730.28
			Under Tol: 47.77 kWh	714,887 kWh &	\$42,110.10
Best Buy Co.	63169-50366	1880 Palm Beach Lakes Bivd	& 19.84% KWD	585 KWD debited	debited
Ocean Properties Ltd.	70876-34924	100 Riverfront Blvd.	5.78%	289	\$2,995,63
Target Corporation	02873-11708	21637 State Road 7 #Target	2.73%	0	\$0.00
Target Corporation	39242-15316	1901 N Congress Ave	4.61%	298	\$3,086.67
Target Corporation	36908-36659	5150 14th St W	2.68%	0	\$0.00
Target Corporation	13854-10566	1200 Linton Blvd # Target	1.73%	0	\$0.00
Target Corporation	42298-19083	13711 S Tamiami Tri #300	4.22%	268	\$2,481.81
Target Corporation	07710-59334	3251 Hollywood Blvd #300	2.02%	0	\$0.00
Target Corporation	10054-45984	1400 Tamiami Trl # Target	3.25%	0	\$0.00
Target Corporation	49909-58540	5350 Fruitville Rd	3.14%	٥	\$0.00
Target Corporation	59543-43371	1271 Tamiami Trl S # Target	3.11%	0	\$0.00

Docket 030623 - Commission Staff Request Question 5.

 $\zeta_{\rm c}$

DUNCAN WATTHOUR AND THERMAL DEMAND METERS

TMS&TMT

TECHNICAL MANUAL

 $-2-$

?

GENERAL INFORMATION

The Duncan Type TMS singlephase and TMT polyphase meters are combination watthour and thermal watt demand meters. With the exception of adding current transformers to the current leads and potential secondaries to the potential stators to supply current and potential for the thermal unit, the construction and circuitry of these meters is basically the same as the corresponding MS and MT meters. The addition of the thermal unit also requires a deeper cover. Polycarbonate covers are standard on alt socket, "K," and "B" type thermal meters. Glass covers are standard on **"A"** base type and the T-9S meters. Standard **MS** and MT watthour registers are used to record the kilowatt hours on their respective meters.

All combination watthour and thermal watt demand types are supplied with a dual range switch. The switching arrangement includes an interlocking feature between the scaleplate and the location of the range changing switch contact screw, which insures that the current transformers are connected correctly for the selected scale. The reset wire in the cover is insulated to electrically isolate it from the meter frame. **A** frame ground strap is also provided on all "S" type thermal meters to give added protection for the meter reader when resetting the maximum pointer.

Both the watthour and thermal are in accordance with ANCl C-12 standards. The watthour section of the TMS and TMT meters conform to ANCl C-12.10 standards and ANCl C-12.5 standards for the thermal unit.

The TMT has all the advantages of the MT; improved stability; linearity and uniformity of adjustment response. SeeTable **B** for range and sensitivity adjustments. See Figures **1** and 2 for location of adjustments. When checking or recalibrating the watthour section of TMS or TMT meters, follow the same procedure **as** used on MS or MT meters.

CHECKING ZERO CALIBRATION **ON TMS AND TMT METERS**

After *a* thermal meter has been de-energized for some time, it may not read exactly zero. *No* adjustment should be made until the meter has been warmed up as follows:

Step **1.** Connect all potential circuits in parallel.

- Step 2.Apply rated voltage only for 2.0 hours. (Current circuit should be disconnected.) The cover must be in place and the maximum (black) pointer must not be in contact with the indicating (red) pointer for this test. .
22. maj
22. maj - Johann J
- Step 3. If the indicating pointer is not indicating zero after the two hour period of potential only, it can be adjusted to zero by turning the zero adjustment, Item 2,

Figure 2, in the + direction (clockwise)
to move the pointer upscale, or in the —
direction (counter-clockwise) to move the pointer downscale.

Care must be taken not to wrap the zero calibration spring, Item 4, Figure 3, around the capstan, Item 5, Figure 3, when making this adjustment.

If the thermal unit has had extensive repair, the

- following procedure for setting zero is as follows: Step **1.** Completely release the zero and full scale adjusting springs, I terns **4** and 2, Figure 3, by turning the zero adjustment screw, Item 2, Figure 2, counter-clockwise, and by turning the full scale adjusting screw, Item 3, Figure 2, clockwise.
	- Step 2. Replace cover and apply rated voltage only (no current) for 2 hours.
	- Step 3. Remove cover. While holding the indicating pointer hub, Item 6, Figure 3, with a %" end wrench, grasp the indicating pointer bracket, Item 7, Figure 3, and rotate the pointer *so* that when the pointer is in its free position, the upper end of the pointer will point to the graduation mark located to the left **of** zero. Replace cover immediately,
		- NOTE: Setting of the indicating pointer must not take more than one minute after cover has been removed.
	- Step **4.** Let meter set for **a** minimum of one more hour with rated voltage only. **This** will be a total of 3 hours with voltage **only.**
	- Step 5. Recheck the indicating pointer setting. If the pointer indication has changed more than the width of the mark **at** the left of zero, rerun Steps **3** and **4** before continuing the test. If the pointer has not changed, remove cover and turn the zero adjusting screw clockwise until the pointer is pointing to the zero graduation mark, Tap meter lightly while making this adjustment. Setting pointer should be done within 20 seconds after cover has been removed.
		- CAUTION: Be certain no current **is** connected to the meter during these five steps. *Any* line current in the circuit during this test will cause an error in the zero setting.

CALIBRATION OF **THERMAL WATT DEMAND** UNIT

Thermal demand meters are defined as lagged demand meters, or those which operate in a manner that requires a specific time for the indication to reach a point corresponding to the value of the applied load. The indicating pointer has an approximate exponential response to the applied load. The value. When controlling the load manually, it is mention to value the value of a larged demand meter is con-
The characteristic of a lagged demand meter is con- encessary time characteristic of a lagged demand meter is con-
sidered to be the time required for the indicating and during the last one-third of the 45 minutes. The sidered to be the time required for the indicating and during the last one-third of the 45 minutes. The soline
state to reach 90% of final indication after a solid is set and occasionally checked to see that it pointer to reach 90% of final indication after a load is set and occasionally checked to see that it
constant load has been suddenly applied. (See is within 2% during the first part of the test, and Cigure **4.)** within the limits as determined **by** the method of

is within 2% during the first part of the test, and

-

Thermal meters are most conveniently tested on gang racks with provision for 5 to 20 meters. Either of two methods of providing a standard reading is recommended. One method involves the **use** of a master thermal demand meter of correct rating, and of known calibration, connected in series with the meters under test. With this method, the meters under test are calibrated to read the same as the master or standard meter.

A second method involves the time-load principle. Here the load is held constant at a predetermined level by reference to a watt-meter, and the meters under test are cafibrated *to* indicate this load correctly.

It is necessary to maintain the load for a minimum of 45 minutes, which is approximately three times the time characteristic described above. The range over which the load may vary during a test is determined by the method of testing used. Where . master meter is used, the load may be allowed to vary over limits as wide as plus or minus 4%, whereas for time-load testing, the load should be controlled within **plus** or minus **1%** of the nominal

test during the last 15 minutes.

Thermal demand meters should always be tested with the covers in place. When the cover **is** removed from the meter, the cooler outside air rushes in and cools the so-called hot element *of* the thermal unit much faster than it does the cold element. This causes a rapid change in the reading of the meter. For this reason, any calibration of the meter must be done quickly after the cover has been removed, preferably within 20 seconds. If other tests *are* to be made, the cover should be replaced **as** soon as possible. If it is desired to recheck **a** calibration point after the cover has been removed and replaced, the present load on the meter must remain constant for a minimum of 45 minutes after replacing the cover before a reading is taken.

The efficiency and accuracy of calibrating thermal demand meters can be improved by the use of test covers that have *3h* " diameter holes located over the zero and full scale calibration adjusting screws, allowing the meter to be calibrated at zero and the calibration point without removing and replacing the cover.

FULL SCALE CALIBRATION

The calibration test point is a point on the scale at which the meter is adjusted to read correctly.

Thermal meters have two adjustments, namely, zero and the full scale adjustment. Normally when making acceptance and periodic checks, they are limited to these two points. However, when desired, additional checks may be made at 50% lagging power factor, and for equality of current circuits.

- NOTE: All errors in registration are figured in % of full scale.
	- Example: A one division error any place on a

100 division scale would be an error of **1 .O%** All external mounting dimensions, terminal arrangements and circuitry on thermal TMS and TMT meters are the same as their respective **MS** and MT watthour meters. See Pages 21 through 30 for wiring diagrams.

The calibration test point can be made at any point from 50% of full scale to 100% full scale. Duncan thermal meters are calibrated during factory calibration at 50% of full scale **KW** for the convenience of using this point on the scale as a comparison for other tests. The cover must be in place and the maximum pointer must be in contact with the indicating pointer for all tests other than zero.

It is possible to test polyphase thermal meters in the shop **on** polyphase loads, *but* the elaborate testing equipment needed for such tests is seldom warranted since singlephase test results can be correlated to polyphase performance. Therefore, polyphase meters are tested singlephase by connecting the potential circuits in parallel and the current circuits in series.

After the zero setting has been completed, the calibration test point can be checked by the following procedure:

Connect the potential circuits in parallel and the current circuits in series.

Suddenly apply a singlephase load at unity power factor equal to the **KW** desired to calibrate or check the meter under test. This load must be held for a minimum of 45 minutes. The accuracy of this load must be held, depending on the method being used for testing, *as* described in **a** previous paragraph.

Calibrate the meter at the KW selected for the calibration point **by** means of the full scale calibration adjusting screw, Item 3, Figure **2.** When adjusting downscale, the indicating pointer should be moved downscale past the calibration points and then adjusted upscale very slowly to the calibration point with the maximum pointer in contact with the indicating pointer. Care must be taken not to wrap the calibration spring, Item 2, Figure 3, around the capstan, Item 3, Figure **3.**

It will not be necessary to recheck the zero setting after the calibration point has been set since the zero and full scale adjustments are independent of each other.

An exception to the above procedure must be made when making the calibration test load or applying a singlephase load to the 3 phase, **4** wire wye, 2 stator meters, i.e., Forms 6S, 7S, and **14s.** These meters have three current circuits, one of which is associated with both potential circuits. When applying a singlephase test load, this current circuit gives 50% full scale reading with only **75%** of full scale current at unity power factor. The other two current circuits are associated with one potential circuit, and each circuit gives 25% full scale reading when energized separately with 75% full scale singlephase current at unity power factor. Therefore, when testing by the singlephase method, apply only 75% of the current for any test point selected.

CHANGING FULL **SCALE** DEMAND **RATING**

To change the full scale demand rating, the following steps must be done. (See Figure 6 for identification of parts.):

- 1. Loosen the two nameplate screws, Item 20, Figure 6, $\frac{1}{2}$ turn and remove the nameplate.
- 2. Remove contact screw, Item 8, Figure 6.
- 3. Loosen the two scaleplate screws, Item 21, Figure 6, a minimum of $\frac{3}{4}$ turns.
- **4.** Remove scaleplate by raising it approximately 1/16" and lift nameplate upward.
- 5.Turn scaleplate over and position it into the

recess in the thermal unit frame. Before tightening the scaleplate screws, orient the scale down and to the right against the screws, Item 21, Figure 6.

- NOTE: Care must be taken to prevent bending the pointers and to keep from trapping the indicating pointer under the scaleplate.
- 6. Remove switch lug, Item 10, Figure 6, from switch block and move to the opposite location so that the contact screw will align with the notch, Item 22, Figure 6, in the nameplate. This will assure that the **KW** rating showing will match the correct thermal current transformer ratio.
- 7. Replace nameplate and tighten screws, Item 20, Figure 6.

The calibration performance of both sides of the scale are checked **on** all meters during factory testing. Therefore, no touch-up or recalibration is required when the scale is reversed for class change.

MAINTENANCE PROCEDURES FOR TMS AND TMT METERS

Replacement *of* Indicating (Red) Pointer

Replacement of the indicating pointer requires the removal of the nameplate and the two 0-80 screws at the lower end of the pointer where it is attached to the indicating pointer bracket. After replacing the new pointer, it will be necessary to check the clearance between the underside of the indicating pointer and the maximum pointer hub, Dim. **"A",** Figure 5. If it is touching, grasp the pointer approximately where it touches the maximum pointer hub and raise it up until it reforms the 90° bend at the lower end of the pointer enough so that it will clear the maximum pointer hub a minimum of **'/64."** Then check to make sure the indicating pointer and the maximum pointer touch only at the flag, Dim. "B," Figure 5, on the maximum pointer and the knife edges of the pointers align with each other and the indicating pointer is approximately **i/32** " above the scaleplate, Dim. "C," Figure 5. Replace the nameplate. There should be approximately V_{32} " clearance between the indicating pointer and the back side of the nameplate, Dim. **"D,"** Figure 5. The meter should be in its operating position when making this adjustment.

Replacement of Maximum (Black) Pointer

To replace the maximum pointer, Item 2, Figure 6, remove the nameplate and the two 4-40 \times $\frac{\gamma_{32}}{2}$ screws, Item 16, Figure 6, holding the grease damping unit in place. Push the maximum pointer to the full scale reading, grasp the maximum pointer at the lower edge of the scaleplate and raise the grease damping unit, Item 3, Figure 6, forward and up. Remove the old pointer by loosening the 0-80 screw in pointer hub. Put new pointer on staff and tighten screw. The underside of the pointer should clear the top of the grease damping unit a minimum of **1/64** ," Dim. "F," Figure 5. To reassemble, push the pointer and grease damping unit under the indicating pointer into the proper mounting place and reassemble the two 4-40 x $\frac{\gamma_{32}}{\gamma_{32}}$ screws. After the unit has been reassembled, check to make sure the end *of* the pointer does not touch the scaleplate over the entire length of the pointer travel, Dim. "E," Figure 5, with the scaleplate firm against its lower stops.

Check to make sure the pointers touch only at the flag, Dim. "B," Figure 5, on the maximum pointer, and the hub on the maximum pointer does not touch the underside of the indicating pointer, Dim. "A," Figure 5.The maximum pointer must be flush with the front surface of the scaleplate or not more than **1/32** " behind the front surface. With the meter in its operating position, the flat portion **of** the upper ends of the pointers must align with each other and must be vertical to the front surface of the scaleplate when the indicating pointer is

against the flag on the maximum pointer.

Replacement of Grease Damping Unit

When replacing the grease damping unit, Item 3, Figure 6, use the same procedure as replacing the maximum pointer.

Replacement of Scaleplate

When ordering replacement scaleplates, be sure to specify the code numbers on the lower left and right hand corners of both sides of the plates. When installing the scaleplates, be sure to position them in the recess in the thermal frame and, before tightening the scaleplate screws, orient the scale down and to the right against the screws, Item 21, Figure 6.

Replacement of Dual Range Switch Block

6. Replacing the dual range switch, Item 18, Figure

- Step 1. Remove the nameplate.
- Step 2. Remove contact screw, Item 8, Figure 6.
- Step 3. Remove scaleplate, Item 4, Figure 6.
- Step 4. Unsolder current transformer secondary leads from the solder lugs on the switch block assembly.
- Step 5. Remove $4-24 \times 4$ self tapping screw, Item 11, Figure 6. When replacing new switch block assembly use reverse procedure.

Replacement of Calibration Springs

Installing zero springs, Item 5, and/or full scale adjusting springs, Item 6, Figure 6.

- Step 1. Attach spring to chain.
- Step 2. Before attaching the spring to the indicating pointer bracket, Item 13, Figure 6, let the spring and chain hang free to remove any twist in the chain. Then attach the spring to the indicating pointer bracket without twisting the chain.

Replacement of Capstan

Replacing capstans, Item 7, Figure 6, zero or full scale calibration chain, Item 23, Figure 6, or cup washer, Item **14,** Figure 6.

- Step 1. Remove the zero or full scale calibration springs, 'Items 5 and 6, Figure 6, from the indicating pointer bracket, Item 13, Figure 6.
- Step 2. Loosen the 2-56 x **3/16** clamping screw, Item 12, Figure 6, on the capstan.
- Step 3. Remove the 5-40 x *9h6* screw, Item 15, Figure 6, and cup washer, Item 14, Figure 6.
	- NOTE: Since the clamping screw distorts the threads on the 5-40 x *9/,6* screw, it is difficult to remove. It may be necessary to use a screwdriver in the slot in the capstan or a pair **of** pliers to hold the capstan while removing the screw. It is recommend-

ed that a new screw and cup washer be used when reassembling the capstan. Care must also be taken not to lose the chain anchor, Item 17, Figure 6. This is a *7/31* " long phosphor bronze wire #29 *8* & S gauge .0112 diameter, located in a hole in the end of the capstan.

the contract of the contract of the second second contract of the contract of

- Step 4. To install new capstan, use reverse procedure as used when removing. The chain and chain anchor wire must be assembled to the capstan before reassembly to the thermal frame.
	- CAUTION: The turning torque of the capstan and the torque of the 2-56 clamping screw is very critical. If the torque is set too low the thermal element may lose the full scale or zero calibration. If the torque is set too high it may collapse the cup washer and make it very difficult to turn the capstan when calibrating the meter. The torque on the capstan should be set with a torque screwdriver, set at 2% to 3% inch pounds. The 2-56 clamping screw should be set at 1.0 to 1% inch pounds.

See Maintenance Note 4, Page 10.

3eplacement of Thermal Element Bearings

If it is necessary to change both front and rear bearings, always keep one bearing in place while changing the other. Do not have both bearings out of the meter at the same time. Keeping one bearing in place will keep the bi-metal coils inside the thermal element centrally located between the heater units.

The following procedure is recommended for changing the bearings.

Step 1. To replace the front bearing, place the meter on bench with the scaleplate facing **UP.**

Step 2. Remove nameplate.

- Step 3, Remove indicating pointer, Item **1,** Figure **1,** by removing the two *0-80* screws from the indicating pointer bracket, Item 13, Figure *6,* at the lower end of the pointer.
- Step **4.** Remove the grease damping assembly, Item 3, Figure 6.
- Step 5. Loosen the 2-56 \times 3/16 clamping screw, Item 12, Figure *6.*
- Step *6.* Remove bearing, Item 9, Figure 6. See Maintenance Note 1, Page 10.

Step **7.** Install new bearing.

CAUTION: Care must be taken when installing the bearing, or the bearing wire may be bent. Place meter on bench with the indicating pointer bracket, Item 13, Figure 6, facing up and to the right. Grasp the indicating pointer bracket with the left hand and guide the bearing wire into the pivot hole in the end of the shaft as the bearing is being screwed into the frame. After the bearing wire has entered the bi-metal shaft. continue to screw the bearing in until there is no end play of the shaft. **As** the bearing is being screwed in, continue to work the shaft in and out with the left hand on the indicating pointer bracket until there is no end play. Then back the bearing out a minimum of % turn to a maximum of $\frac{1}{2}$ turn. This will give an end play of *.006* to .0125. See Maintenance Note 2, Page 10.

-

- Step 8. Tighten the 2-56 clamping screw, Item 12, Figure 5, with a torque screwdriver set at 0.5 to 1.0 inch pounds. See Maintenance Note 1, Page 10. If the front bearing is the only bearing to be replaced, reassembly the grease damping assembly, indicating pointer and nameplate. If the rear bearing is to be replaced, continue on to Step 9.
- Step 9 Unsolder the potential transformer secondary leads, Item 1, Figure *3,* from the thermal unit heaters. (Both potential transformers on polyphase meters).
- Step 10. Remove contact screw, Item 8, Figure 6.
- Step 11, Remove scaleplate, Item 4, Figure **6.**
- Step 12. Remove the $4-24 \times 15$ self-tapping screw. Item 11, Figure 6. This will free the range changing switch so that the current transformer leads will not have to be unsoldered.
- Step 13. Remove the two $6-32 \times \frac{1}{2}$ thermal unit mounting screws, Item 19, Figure 6.
- Step **74.** Loosen the rear 2-56 clamping screw, same as Item 12, Figure 6, except on rear of thermal unit.
- Step 15. Remove rear bearing, ltem 9, Figure 6. See Maintenance Note 1, Page 10.
- Step 16. To replace the rear bearing, place the thermal unit on bench with indicating pointer bracket up and to the left, and quide the bearing wire into the bi-metal shaft by grasping the indicating pointer bracket with the left hand while screwing the bearing into the frame. Using the same method for checking the end play *of* the bi-metal shaft as used when inserting the front bearing, screw the bearing in until no end play exists. Then back the bearing out a minimum of $\frac{1}{4}$ turn to a maximum of $\frac{1}{2}$ turn.

See Maintenance Note 1, Page 10.

- Step **17.** Tighten the 2-56 clamping screw against : the bearing with a torque screwdriver set from *0.5* to **1.0** inch pounds.
- Step **18.** Put thermal unit back on meter frame and reassemble **all** other parts in reverse order as used when disassembling the unit.

See Maintenance Note **4,** Page 10.

MAINTENANCE NOTES

- NOTE **1:** When the 2-56 clamping screw, Item 12, Figure 6, **is** tightened against the bearing body, it tends to flatten the threads **on** the bearing in that area. In some cases this may cause some difficulty in removing the bearing and may require retapping the 5-40 bearing hole in the frame, after removal **of** the bearing.
- NOTE 2: The bearing screw has a 40 pitch thread. This **is** the same as used on micrometers. Therefore, one turn of the screw will give .025 movement of the screw. This makes it easy to very accurately control the amount of end play of the bi-metal shaft.
- NOTE 3: **If** the thermal element, indicating pointer bracket, or current transformer needs to be replaced, the meter will have to be returned to the Duncan Electric Company for repair.
- NOTE **4:** Replacing the thermal element bearings, calibration chain, or capstans is very critical to the operation of the thermal section of the meter. If proper tools are not available for these operations, it is recommended that the meter be returned to Duncan Electric Company for these replacements and adjustments.

t These parts include mounting screws.

t t **Give** serial number of meter when ordering calibration springs.

Specify code numbers and/or letters in lower right and left-hand corners, on front and back of scaleplate.

*. Assembly of capstan includes capstan, capstan clamping screw. capstan mounting screw, cup washer, chain, and anchor. The anchor is a *7h~"* long phosphor bronze wire **(#29** B&S gauge), and for shipment is held in position with a piece of tape. Remove tape and clean end of capstan with naptha, alcohol, or other solvent before assembling capstan to meter.

 \mathbb{Q}^*

PAD LOCK RESET **STANDARD** RESET

TMT-12S TMT-5S 27962 58123-2			Form	Number
	TMS-2S			58133-1
	TMT-6S 28006 TMS-3S	58131-2	TMT-14S	58127-1

the contract of the company of the contract of

.
Tanàna amin'ny faritr'i Nor

~-

K_h values for other voltage ratings are proportional.

Watthour Constants, Gear, and Register Ratios TMS and TMT

The watthour constant Kh **is** defined as watthours per revolution of the disk, and can be calculated for single stator meters as follows:

 K_{H} = Rated Voltage x Meter TA Rating Base Speed (Rev./hr. of the disk)

The base speed of Duncan MS and TMS single stator TA 30 meters is 16²/₃ RPM or 1000 RPH. The base speed for Duncan type "K" single stator TA 50 meters is **13%** RPM or **833%** RPH.

Example: TMS-3S, 120V, TA 2.5

$$
\kappa_h = \frac{120 \times 2.5}{1000} = 0.3
$$

is **6"1/ls** RPM or **4161/,** RPH. The base speed of Duncan **MT** and TMT 2 stator TA 30 meters is $8\frac{1}{3}$ RPM or 500 RPH. The base speed for Duncan type "K," 2 stator TA 50 meters

culated as follows: The Kh for 2 stator MT and TMT meters **iscal-**

$$
K_h = \frac{\text{*Rated Voltage} \times \text{Meter TA Rating} \times \text{*2}}{\text{Base Speed (Rev./hr. of the disk)}}
$$

*For Duncan two stator, 3 phase 4 wire wye meters, use a multiplier of 3 instead of 2. Use 240 volts when figuring the Kh on 277V wye meters.

Example: **TMT-l4S,** 277V, TA **30**

$$
K_h = \frac{240V \times 30 \times 3}{500} = 43.2
$$

Register Ratio Calculations

The units **dial** on all watthour registers records 10 kilowatt hours, or 10,000 watthours per revolution. The total gear ratio **(Rg)** between the meter spindle **and** the first dial shaft **IS** given **by:**

$$
R_g = \frac{10,000}{K_h}
$$

In all Duncan meters, this reduction is taken in two stages. The first reduction *is* between the spindle worm and the worm wheel. On Duncan type MS and TMS meters this reduction is a 100/1 ratio. On Duncan type MT and TMT meters this reduction **is** a 50/1 ratio. Hence, the register ratio (Rr) can be calculated **as** follows:

i

$$
R_r = \frac{Rq}{100}
$$
 for single stator TMS meters

$$
R_{\rm r} = \frac{R_{\rm g}}{50}
$$
 for 2 and 3 stator TMT meters

Example for single stator TMS-2S, **240V,** TA 30, Kh 7.2:

$$
R_g = \frac{10,000}{7.2}
$$
 1388% $R_r = \frac{1388\%}{100} = 13\%$

Example for a 2 stator TMT-l4S, **120V,** TA 30, **4** wire wye meter, Kh 2 1.6:

$$
R_g = \frac{10,000}{21.6} = 462^2\%_7 \quad R_r = \frac{462^2\%_7}{50} = 9\%_7
$$

Dial Multipliers **Kr**

Some registers require **a** dial multiplier (Kr) to prevent the register from starting to repeat during a bitting period. When these constants are necessary, they are printed on the register face, and the gear ratio and register ratio are calculated as follows:

.
. .

Single stator MS and TMS meters:

$$
R_g = \frac{10,000 \times K_r}{K_h} \qquad R_r = \frac{R_g}{100}
$$

Two or three stator MT and TMT meters:

$$
R_g = \frac{10,000 \times K_r}{K_h} \qquad R_r = \frac{R_g}{50}
$$

Constants Used With Instrument Transformers

When combination watthour and thermal watt demand meters are to be used with instrument transformers, they usually *carry* secondary **KW** ratings and register ratios. These secondary **KW** readings and watthour register readings must be multiplied by the product **of** the instrument transformer ratios (T.F.) to determine the maximum demand of the load and kilowatthours. When meters are to be used with instrument transformers, the primary disk constant (KhP) is calculated by multi-
plying the secondary disk constant (KhS) by
the T.F.
Example:
 $T.F. = \frac{2400V}{120V} \times \frac{400A}{5A} = \frac{20}{1} \times \frac{80}{1} = 1600$ plying the secondary disk constant (KhS) by the T.F.

Example:

T.F. =
$$
\frac{2400V}{120V}
$$
 x $\frac{400A}{5A} = \frac{20}{1} \times \frac{80}{1} = 1600$

DUNCAN **SOCKET TYPE** METERS **COR RESPOND I NG TO STANDARD FORM DESIGNATIONS**

ļ. Meters have provision **for** three additional terminals for contact device connections **by** request

I 1

 $\frac{1}{2}$

سمعة من المعدن المعد
المعدن المعدن الم

- **17** -

TYPE TMT "B"

 $-18-$

 \sim \sim

 $\label{eq:3.1} \begin{split} \text{where } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{ is the same number of times } \mathbf{q} \text{$

 \mathbf{r}

3.777
MAX.

 \sim

 \sim - \sim

1

I

INTERPRETATION OF WRONG TEST RESULTS FOR TMS & TMT METERS

المستحدث

 \overline{a}

 \sim

المستنا

وستعدد والمستردة

ANDREW CO.

 $\label{eq:1} \begin{aligned} \mathcal{L}^{(1)}(x) &= \mathcal{L}^{(1)}(x) + \mathcal{L}^{(2)}(x) + \mathcal{L}^{(1)}(x) + \mathcal{L}^{(2)}(x$

 \mathcal{N}^{\pm}

 $\mathcal{L} \rightarrow \mathcal{L}$

 $-24-$

l.

 -25

 \sim .

 \mathbf{r}

 \mathcal{L}^{max}

 $\frac{1}{2}$, $\frac{1}{2}$

 $-27-$

 $\overline{}$

 \mathcal{L}

ù,

 $\mathcal{L}^{\mathcal{A}}$

 $\label{eq:reduced} \begin{split} \mathcal{L}_{\text{in}}(\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{out}},\mathcal{L}_{\text{$

റവ

 \sim .

 $\label{eq:1} \begin{array}{c} \mathcal{L}_{\text{max}}(\mathcal{L}_{\text{max}}) \\ \mathcal{L}_{\text{max}}(\mathcal{L}_{\text{max}}) \end{array}$

 $\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}$

 $\overline{}$

 \mathbb{R}^2

 \sim

01/18/04

Thermal Meters

In early 2002, consultant (George Brown), representing FPL customer, brought to FPL's attention that a 1V meter's demand registration *was* over-registering due to temperature changes resulting froni sunlight. FPL's tests on meter proved customer's allcgarions were correct. Refund was made.

50 meter sample on I V meters and 100 meter sample of 7 other thermal meter types indicated that no other meters demonstrated sensitivity to sunliglit. However, sample did indicate that the **1** V thermal meter population (3900 meters) was outside of allowed tolerance and needed to be removed and replaced.

Replacement of all 1V meters initiated in 10/02, with all 1V's to be removed by 01/03 and tested by 3/03. **No** back-billing for meters under-registering. Kehnds io customers to be made for meters over-registering - back to point of meter failure or 1 year if not determinable (per FPSC rules). Customers with multiple accounts (Target, Dillards, etc.) will net over-billings with under-billings. 1V plan presented *to* FPSC Staff.

Sample testing on each of the 7 other thermal meters initiated. Results of 4Nmeter sample available in 10/02. Sample passed but nearing maximum allowed defective rate. Decision made to remove **4N** population (4600 meters). Because population passed, no testing of meters removed to be done.

In late 2002, other sample results become available. All pass except 1U meter **(1** 1,000 meters). **4L** (24,900 meters) and **4J** (17,400) nearing nuximum allowed defective rate. *2* more 1U samples are initiated to validate first sample that failed.

In 1/03, 2 additional 1U samples pass. However, replacement of 1U meters initiated – half to be replaced in 2003 and the remainder in 2004.

In $3/03$, changes were made to the 1V meter refund process to insure all customers are fairly treated – Meters tested at 40% of full scale and over-registering to be re-tested at 80% of full scale registration (700 meters). Also, customers with meters over-registering out of tolerance will get refunds based on the higher of the meter test error or the difference in their consumption since the new meter was installed.

1V meter test results show 85% within tolerance, 10% of meters under-registering (demand and kWh) and 5% over-registering (demand).

George Brown's clients include accounts such as Target, J C Penney, Dillards, Walmart, Best Buy, Home Depot, Kash n Karry, and Food Lion.

FPSC Staff currently reviewing FPL and George Brown positions on issues – key issue is determining when meter error occurred. George Brown position - error occurred when meter was set or last tested because meters were mis-calibrated by FPL - this would result in multiple year refunds. FPL position - error point cannot be determined since error occurred gradually - therefore, rehnd should be for **1** year. FPSC docket opened in July 2003 - no schedule for docket yet.

Palm Beach Post reporter, contacted by George Brown, prints several recent articles on 1V meter issues. Questions raised about under-registering meters and "line losses". Also initiates dialogue with Public Counsel, who has now requested information on potential fuel clause impacts.

2003 sample results *to* be available 9/03. Results will influence removal of remaining thermal meter types.

SWAT team initiated to identify all impacts and make recommendations.

 \sim

 $\sqrt{2}$

Zero Adjustment Test

- 1. Energize meters to be tested on back of test board.
- *2.* Load all meters of the same form and voltage onto one bank of sockets.
- *3.* Set switches to match the form and voltage of meters for that bank
- **4.** Run test for at least 2 hours.

OBSERVATION POINTS

- A. Be sure friction pointer is not touching the pusher pointer for this check.
- B. The cover should be in place during this check.
- *5.* Repair and/or adjust the zero position hand if necessary.

Load Calibration

- 1. Load meters to be tested on the front of the test board.
	- A. Meter to be tested MUST be of the same form and voltage rating. Unused sockets must be jumpered across.
	- B. Covers should be on the meter for-this test.
- 2. Set all switches and controls to positions shown in the "Thermal Test Board Setup Data Sheet" for form of meter under test.
	- A. All variable controls, such as the current control should be fully counter-clockwise.
- **3.** Switch the voltage circuit breaker to ON and adjust the TEST VOLTAGE FINE ADJUST control for a reading of 120 volts on the voltage panel meter.
- **4.** Switch the current circuit breaker to ON and slowly bring up the TEST AMPERES COURSE ADJUST to approximately the reading needed on the ampere panel meter.
- *5.* Adjust the TEST AMPERES FINE ADJUST to bring the current to the amperes called for on the setup data sheet.

- 6. The PHASE ANGLE control should always be set to 0 degrees. Adjust the PHASE ANGLE FINE ADJUST control until the POWER FACTOR panel The FriASE ANGE
PHASE ANGLE FI!
meter reads 1.0%.
- *7.* Run load test for at least one hour.

OBSERVATION POINTS

- A. The standard meter on the board should read zero before starting the test.
- B. Check that all reset arms are out of the way of the pusher pointers.
- C. Check all meters under test for rotation of the disk.
- D. Check all meters for free movement of the demand hand (friction pointer).
- E. Periodically check panel instruments for variations. Correct any variations as necessary.
- 8. At the end of 1 hour of load test, calculate the percent error as shown on the "Thermal Test Board Setup Data Sheet" for each meter tested.
- 9. If the calculated percent error is greater than +/- **4** percent, adjust the meter using the full load adjustment and adjust to 0% error.
	- A. This adjustment must be made when the meter is fully energized at test power, and the friction pointer in contact with the pusher pointer.
	- B. The meter cover should only be removed long enough to adjust the meter and should be replaced as soon as possible.
- 10. If a meter has been adjusted, the test board should be left energized, with a stable load, for approximately IO minutes, to check for proper calibration.
- **¹**I. After all meters are calculated to be as close to 0% error as possible, the current should be slowly lowered to zero.
- 12. The voltage and current circuit breakers should be turned OFF.
- **13.** The meters should not be removed from the test socket for the length of cool down time shown the chart below:

Distribution Meters

 $\bar{\mathbf{R}}$

OBSERVATION POINTS

- **A.** The back of the test board may be loaded with additional meters at any time during the load calibration test.
- B. Self-contained meters should always be allowed to cool down before removal to prevent air line rupture.

#.&om **Meter Testing** Operations

THERMAL TEST BOARD PROCEDURES

Zero Adjustment Test .-

- 1. Energize meters to be tested on back of test board.
- 2. Load all meters of the same form and voltage onto one bank of sockets.
- 3. Set switches to match the form and voltage of meters for that bank.
- **4.** Run test for at least 2 hours.

OBSERVATION POINTS

- **A.** Be sure friction pointer is not touching the pusher pointer for this check.
- B. The cover should be in place during this check.
- *5.* Repair and / or adjust the zero position hand if necessary.

Load Calibration

- 1. Load meters to be tested on the front of the test board.
	- **A.** Meters to be tested MUST be of the same form and voltage rating. across. Unused sockets must be jumpered
	- B. Covers should be on the meter for this test.
- *2.* Set all switches and controls to positions shown in the "Thermal Test Board Setup Data" sheet for **form** of meter under test.
	- **A. All** variable controls, such as the current control should be fully counter-clockwise.
- *3.* Switch the voltage circuit breaker to ON and adjust the TEST VOLTAGE FINE ADJUST control **for** a reading **of** 120 volts on the voltage panel meter.

- 4. Switch the current circuit breaker to ON and slow up the TEST AMPERES COURSE ADJUST to approximate1 reading needed on the ampere panel meter.
- *5.* Adjust the TEST AMPERES FINE ADJUST to bring the current to the amperes called for on the setup data sheet.
- 6. The PHASE ANGLE control should always be set to 0 degrees. Adjust the PHASE ANGLE FINE ADJUST control until the POWER FACTOR panel meter reads *1.04.*
- *7.* Run load test for at least one hour.

OBSERVATION POINTS

 \sim

- **A.** The standard meter on the board should read zero before starting the test.
- B. Check that all reset arms are out of the way of the pusher pointers.
- C. Check all meters under test for rotation of the disk.
- D. Check all meters for free movement of the demand hand (friction pointer).
- E. Periodically check panel instruments for variations. Correct any variations as necessary.
- *8.* At the end of 1 hour of load test, calculate the percent error as shown on the "Thermal Test Board Setup Data" sheet for each meter tested.
- *9.* If the calculated percent error is greater than $+$ 4 percent, adjust the meter using the full load adjustment and adjust to 0% error.
	- **A.** This adjustment must be made when the meter is fully energized at test power, and the friction pointer in contact with the pusher pointer.
	- B. The meter cover should only be removed long enough to adjust the meter and should be replaced as soon as possible.
- 10. If a meter has been adjusted, the test board should be left energized, with a stable load, for approximately 10 minutes, to check for proper calibration.
- 11. After **all** meters are calculated to be as close to 0% error as possible, the current should be slowly lowered to zero.
- 12. The voltage and current circuit breakers should be turned OFF.
- 13. The meters should not be removed from the test socket for the length of cool down time shown in the chart below;

Instrument transformer rated - ⁰ Self contained (100 amps) - 10 minutes Self contained (200 amps) - 15 minutes

OBSERVATION POINTS

- **A.** The back of the test board may **be** loaded with additional meters at any time during the load calibration test.
- B. Self contained meters should always be allowed to cool down before removal to prevent air line rupture.

 \downarrow

Draft for our meeting this afternoon:

Possible "script" for the approximate 60 meters that cannot be located (most of these -50 or so were never tested, while the others were due for re-test at 80%):

Remind customer of 1V meter removal plan (affected customers were contacted by letter and/or phone call in latter part of 2002)

Inform them of of our overall testing results for the approxjmate 3900 meters - 85% tested within tolerance, 11% were under-registering out of tolerance and 5% were over-registering out of tolerance

Unfortunately, their particular 1V meter that was removed cannot be located

In order to appropriately remedy this situation, here's how we're handling the misplaced meters (We should have for each account, prior to the contact, the customer's specifics so that the different scenarios can **be** discussed with them:

All affected customers will be receiving a 1 year refund

(1) For those meters without any meter test results -

FPL will provide a 1 year refund using either **4%** or the % change, new vs. old meter, whichever provides the greatest refund

(2) For those meters with an initial meter test result that over-registered (>100%) and no re-test meter result (14 accounts) -

FPL will provide a 1 year refund using 4%, the initial meter test result, or the % change, new vs. old meter, whichever provides the greatest refund

Note - For those meters that were eligible for a re-test we may need to explain that process (this could **be** somewhat confusing though)

000162 TDM

Customer Meetings -_

Purpose of Meetine (Chuck)

Not to argue positions and issues - ensure customers' understanding of our position/legal process Where we are in FPUGB negotiations FPL's refund determination process Staff Rec./Applicable FPSC rules - meter errors and refunds Docket process What's at **risk?**

Negotiations Status (Chuck)

FPL Offer vs. George Brown claim Keason(s) for difference

Staff Rec. / **Applicable FPSC Rules (Dave)**

Staff Recommendation Summary Determination of refund time period $(25-6.103(1) - 1$ year unless error cause and date known Determination of refund % error (Rule 25-6.103(3) – error to be based upon meter test Slow, partially, and non-registering meters $(25-6.103(2)(a,b,c) -$ allows back-billing for 12 months

FPL Refund Determination Process (Chuck)

Re-test at 80% - use higher of two tests Higher of meter test % or usage - new meter vs. old No back-billing,' but will net for multi-account customers

Look at each eligible account on it's own Determine point meter failed, if possible Review usage since meter was replaced vs. previous year(s) usage If usage is similar to previous years - 1-year refund, if not, refund adjusted Use higher of meter test error or actual % difference - new meter usage vs. previous year(s) usage

FPSC Docket Process (Dave)

Staff recommendation Agenda Conference PAA Order Protest/Request for hearing Discoveryldepositiom Testimony Hearing Briefs Staff Recommendation Agenda Conference Appeals

What's at stake? (Chuck)

In certain instances, FPL has gone beyond rules in its offer (> of meter test/actual usage; multiple years vs. 1 year If litigated, FPL will revert back to rules Time period for litigation/appeals - could be years (all other customers have received refunds)

-

Why is FPL's process for handling 1U meters different than the 1V meter process? Unlike the IV meter population, which failed, the **1U** population passed.

KINGS POINT KWD

I. 1V Meter Issues:

What is the appropriate % of full-scale registration to be used for testing the 1V meters?

Per Rule 25-6.052(2)(a) - "when tested at any point between **25%** and 100% of full scale value"

Per FPL IV Meter Process - all low scale meters were tested **at** 80% and **all** high scale meters originally tested at 40%; and over-registering **were** re-tested at 80%.

When a meter tests out of tolerance, what is the appropriate % error to be used?

Per Rule **25-6.103(3)** - "when a meter is found to be in error in excess of the prescribed limits, the figure to be used for calculating the amount of refund or charge in (1) or **(2)(b)** above shall be that % *of* error as determined **by** the test"

Per FPL 1V Meter Process - Refunds based on the meter test result or the difference in customers usage before and after the new meter **was** set, whichever provides the customer with the greatest benefit

What are the conditions that must be satisfied to provide a refund greater than 1 year?

Per Rule 25-6.103(1) $\frac{1}{2}$ the period since the last test, said $\frac{1}{2}$ period should not exceed 12 months; except that if it can be shown that the error was due to some cause, the date **of** which can be fixed, the overcharge shall be computed back to but not beyond **such** date based upon available records

Per FPL 1V Meter Process - Same as Rule 25-6.103(1)

II. 1V Meter Discussion Items:

Overview *of* FPUGeorge Brown's methodology to determine eligibility for /calculation of refunds (contrast approaches - settlement vs. FPSC rules)

FPL Methodology - Compared new electronic demand readings *to* similar months in previous years to determine if error could be identified; if not, was there a materiallconsistent difference in the "new" and **"old"** demands? if *so,* offered refund back over that period. Used higher of meter test results or "new **vs.** old" readings; used average difference for affected years; if change in demand affected rate class, used appropriate rate class to compute re-billings

Account by account review of IV meter complaints (if desirable by Staff)

Chuck to handle (hand **out(s)**

II. 1U Meter Issues:

What is FPL's process for handling 1U meters (as well as 4N meters)?

FPL has initiated an active retirement plan for the 1 **U** meters. Once these **meters are** removed, FPL will retain these meters for **at** least **six** months. If a **custor** they **would** like their meter tested, **FPL** will comply with the request, con **25-6.059.** Refunds and backbillings will be determined consistent with **R**

..

000161 TDM

Table of contents

 \mathbf{r}_{max} and \mathbf{r}_{max}

Introduction **Page 3** Why the term demand _.Page **⁴** Why maximum demand metering................................. Page 6 What is maximum demand metering Page 6 What ... is demand Page *⁵*

Mechanical Demand Register Page **¹⁰**

History of demandPage 8

Thermal Demand Metering. Page 32

Questions and Answers . Page 52

000067 TDM

 \bullet

Introduction

 \mathbf{i}

In recent months, much information has been distributed about demand meters. In the enthusiastic promotion of **a** specfiic product, certain facts have been distorted.

As an example of the misconceptions that arise as a result of promoting one type against the other, one manufacturer announced some time ago that the company was discontinuing its thermal meter because it was inferior to the mcchanical demand register. The facts are, this thermal meter never earned more than five percent of the singlephase thermal demand meter market. In addition, the basic design could not be adapted to the manufacture of a polyphase meter. Obviously, the real reason for discontinuing this thermal meter **was** its limited application and lack of acceptance.

Sangamo, as the largest manufacturer of demand meters and *a* pioneer of both mechanical registers and thermal meters, is best qualified to present a completely unbiased story on both types **of** instruments.

The important reason for reviewing facts about demand metering is graphically presented in Figure **A-1.** The graph illustrates the increasing use of demand metering on a national scale. The upward trend indicates that demand metering is becoming a more significant factor in the operation of an electric utility. In addition to the obvious effect on revenue, the increase in demand metering has broadened the responsibilities of the meter department.

Figure A-1 I **TREND OF TOTAL BILLING DEMAND**

Both mechanical demand registers and thermal demand meters are used for *both* singlephase and polyphase services. Which type of meter should be used, will depend on **a** number of factors; including test facilities, testing schedules, geographical nature of the electric system, and many others. According to recent industry figures, (Figure **A-2)** the ratio of thermal to mechanical meters measuring singlephase loads is approximately 3 to 1. On the other hand, the ratio of mechanical to thermal meters measuring polyphase loads is approximately 3 to 1.

Figure **A-2 PROPORTIONAL APPLICATION OF DEMAND 'METERS (APPROX.1**

The purpose of this presentation is to review the merits and limitations of both thermal and mechanical demand meters so that utilities will have factual information on which to base decisions about demand metering.

Why the term "demand"?

Before discussing demand metering, it seems the first question to ask is "why the term demand?". If psychology has a place in rate making-and it certainly does-then the word "demand" should never have been admitted to the electric utility vocabulary. The term "demand" is applicable here only in the sense that the customer demands that the utility furnish *as* much power as required by the electrical equipment he **may** use at any one time. Although demand charges are justified from the standpoint of economics, the utility customer often fails to realize that the charge is based on his demands upon the utility's services.

Although it may be an oversimplification of the problem, **a** first suggestion would be to drop the word "demand" and substitute in its place **a** logical and less offensive word such **as** "capacity," "power," or "load value". The negative aspects of the term "demand" tend to form a psychological barrier to the acceptance of this **form** of billing. **A** positive approach might achieve quicker understanding and acceptance by the utility's customers.

Although the Sangamo Electric Company suggests that the **use of** the term "demand" be discontinued, it will be used in this brochure because it **is** an expression that already has industry wide acceptance.

000069 TDM

What is **demand?**

Energies expended were equal: but the demands on the "equipment" were different.

Kilowatt-hours, or energy, can be defined as the electric load expressed in watts, times the number of hours it is used. A kilowatt-hour (kwh) can consist of a 1000 watt load, connected for one hour, or a 100 watt load connected for 10 hours, etc.

Kw demand is defined as the electric load averaged over a specified interval of time. The demand interval during which the load is averaged may be any length of time, but is usually **15** or 30 minutes. Instead of averaging demand over a time interval, it could be measured instantaneously. This, however, would not. result in a realistic basis for billing. The utility does not want to penalize the customer for instantaneous peaks created by the starting load of a motor, etc., as they have little effect on the overall electric system.

This brochure is primarily concerned with kw demand measurement because it is the most common form of demand billing. However, demand can also be measured in terms of kva and kvar.

The relationship between energy and kw demand is shown in Figure A-3, which represents power versus time of the load consumed by a customer during demand intervals. The dotted line shows the electric load (power) being used by the customer at any time. The solid line indicates demand or the average electric load during each interval. The area under the dotted line (power) represents the energy or kilowatt-hours used. Note that in any one time interval, the area under the solid line is equal to the area under the dotted line. Since energy is the product of power and time, either area represents the energy (kilowatthours) consumed in the interval. The fact that the areas are equal shows that the demand **for** the interval is that average value of power which will account **for** the same consumption of energy (kilowatt-hours) as the actual power.

0 (IS1 INTERVAL) 15 (ZND INTERVAL] 30 (3RD INTERVAL) 45 (4TH INTERVAL) 60 TIME IN **MINUTES**

000070 TDM

t

Why maximum **demand metering?**

Demand metering, as **a** method of determining customer billing, has become an accepted practice for a number of reasons. According to industry figures, capital investment represents **2/3** of the utility's cost to serve its customers.

To a **large** extent, the maximum demand of the operating system determines the utilities capital investment. The utility should be reimbursed by the customer at a rate related to the capital investment required to provide service. Another reason for demand metering is that it permits customers *to* be billed on **a** more equitable basis.

For example, consider two types of customers (figure **A-4)** who we an equal number of kwh each day. Customer **A** uses electric energy 24 hours a day and Customer B uses electric energy 8 hours per day.

Customer **A:** The customer's load is made almost entirely of synchronous motor-driven pumps which operate at rated load continuously. *24* hr. x 2 **kw** = **48** kwh.

Customer B: This customer uses the same number of kilowatt-hours **as** Customer A, but they are consumed in an 8-hour period. $8 \text{ hr. } x \cdot 6 \text{ kw} = 48 \text{ kWh.}$

Customer B requires the utility to have generating and distribution capacity equal to the ratio of 6/2 times the capacity required to supply Customer **A.** This is 3 times the capital investment that was required to serve Customer **A,** and Customer B should be billed for this extra investment.

a

What is maximum demand metering?

As described, demand metering is the measurement of average power requirements in a time interval. Consequently, demand billing consists of **measur**ing a customer's maximum average load during any demand interval of the billing period. There are two methods for obtaining this maximum demand-mechanical and thermal.

The mechanical method of measuring demand can be accomplished *-by* the direct counting of watthour disk revolutions with a mechanical register during a mechanically timed intervaI. (figure **A-5)**

-and by counting revolutions through impulses which are totalized dur**ing a** mechanically timed interval on **a** separate indicating or chart type meter such **as** the Digital Demand Recorder. (figure **A-6)**

000071 TDM

Figure A-6 DIGITAL DEMAND RECORDER

The thermal method measures demand through an arrangement of bimetal coils and electrical heaters, having an inherent time interval based upon the heating effect of the load. Thermal demand measurement is accomplished.

- -by indicating demand meters with a maximum demand pointer (figure A-7)
- -by graphic demand meters which record demand readings on a chart **(figure A-8)**

WRIGHT DEMAND METER

Figure A-9

One of the first meters known to be used by man was the Calculus. This was a device used in early Roman days to determine the fare for a taxi ride. By a mechanical arrangement connected to the axle of a cart, pebbles were dropped in a jar. At the end of the ride the pebbles **were** counted to determine the revolutions of the **axle** and, in turn, the bill. This early meter is analogous to our present energy measuring meter, in that it measured units of distance while the watthour meter measures units **of** energy. Demand measurement, and charges, might **have** been applied if the Calculus indicated the maximum speed at which the cart traveled, **as** demanded by the passenger.

 $\mathbf{F}^{(i)}$

我

The probable beginning of electrical demand measurement was the Wright Demand Meter (Figure **A-9)** which was patented in 1897. This meter used glass tubes, air, liquid, and *a* heating element to measure demand. Current caused an expansion of air which, in turn, forced liquid into *a* tube which had **a** scale mounted behind it. **The** more current used, the higher the level of liquid, hence **a** higher demand reading.

AT KI N *S 0* N -S C **HA** TTN E K **M** ET E **^R**

The Atkinson- Schattner maximum demand meter was another early demand device (figure A-10). The meter worked on an electromagnetic principle. Current pulled the core into the solenoid which caused the balls to fall away from the scaled tube as the load increased.

In the early *TO'S,* Sangamo offered to the industry the HM mechanical demand register and from this evolved Sangamo's precise mechanical demand registers of today.

Simultaneously, another development was taking place in the demand measurement field. In Canada, through the developments of Mr. P. M. Lincoln, the first thermal demand meter was offered in 1920. In 1925, Mr. Lincoln became associated with Sangamo Limited in Canada for the purpose of developing a combination kwh and kw meter. Today the thermal type meter is the standard demand meter in Canada.

In 1928. **the** Lincoln Meter Company, Inc., **was** organized in the United States. The Sangamo Electric Company purchased all of the outstanding stock of the Lincoln Meter Company in 1940. Since its introduction, the Lincoln (thermal) meter has also firmly established itself in this country as a highly reliable demand billing instrument.

This introduction to "The Facts about Demand Metering" has reviewed some general aspects of the subject. The following sections on mechanical registers and thermal meters are objectively presented to give complete, specific, and unbiased information on both types. Detailed comparative information will be given on:

a

Figure **A-10**

- 1. Principles of Operation
- **2.** Design
- **3.** Important Construction Features
- **4.** "Theoretical" Accuracy
- **5.** "Field" Accuracy
- *6.* Maintenance
- 7. Shop and Field Testing

The mechanical demand register is an instrument which is used in place **of** the standard kilowatthour register to measure kilowatt demand in addition to kilowatthours. **As** its name implies, the mechanical demand register obtains load information through mechanical gearing from the watthour meter disk.

The complete register is made up of the gearing associated with the kwh dials and three mechanisms that operate together to measure maximum average **kw** load in a demand interval. These three mechanisms are :

- **a.** Pusher Arm
- b. Interval Timing
- c. Pusher Arm Reset

After explaining the basic function of the mechanical register, each of the three demand mechanisms will be explained in detail. They will then be combined to show the complete operation of a mechanical demand register.

000075 TDM

¥.

Basic operation

The watthour meter disk turns at a speed proportional to the load. When a mechanical demand register is attached to **a** watthour meter, the **kwh** gear train (yellow-Figure B-Z), similar to the gear train of a standard watthour meter register, records the revolutions of the disk on kwh dials. The same information is fed through the pusher arm gear train (red) to the pusher arm of the demand register and it accordingly moves the maximum demand pointer up scale in proportion to the speed of the disk.

The pusher arm, because it is geared directly to the **kwh** train, would actually drive the maximum demand pointer to record **kwh** if the arm did not reset to zero **at** the end of each time interval. When the pusher arm resets, the maximum demand pointer remains at the maximum indication.

The motor driven timing gearing (white), at specified intervals, signals the pusher arm to stop recording **kwh,** reset to **zero,** and start over. This action is accomplished by the pusher arm reset mechanism (orange). Since the pusher arm is indicating kwh **per** time interval, the scale **is** marked in **kw.**

 \mathbf{F}^{max}

 \mathcal{L}^{max}

Figure **8-3** I **KWH GEARING**

Pusher arm operation

As previously described, the pusher arm *is* geared to the watthour meter disk through the same wormwheel and shaft **as** the kwh dials (Figure B-3). Consequently, the pusher arm would be expected to record kwh if it were not for the reset action. The sequence of pusher arm gearing **(red)** is **shown** in Figure B-4.

The relationship between the kwh dials and the pusher arm indication can be illustrated by the following example and table. Assume that **a 15** minute interval demand register is mcasuring a steady load of 16 kw. After the first **15** minutes **(1/4** hr), the **kwh** dials will indicate **4** kwh. Assuming the pusher arm does not have a reset action, it wiIl **also** indicate **4 kwh** in *15* minutes **or** 16 kwh in one hour. However, since the pusher arm is reset every **15** minutes. it is scaled to indidemand register is measuring a steady load of 16 kw. After the hrs $(1/4 \text{ hr})$, the kwh dials will indicate 4 kwh. Assuming the pusher a have a reset action, it will also indicate 4 kwh in 15 minutes or 16 hour. However,

 \mathbb{S}^n

€

To further clzrify tlie relationship between kwh and kw in **a** dernand register, the following table shows pusher arm indication for 15, 30 and 60 minute interval registers measuring: a 10 **kw** load.

In each register, thc pusher arm will have pushed the maximum demand pointer to a scale indication of 16 kw.

Important characteristics of the pusher arm mechanism include:

-low friction gears and bearings,

- -concentric location of the pusher arm and maximum demand pointer hubs with respect to the arc of the demand scale,
- -constant friction for maintaining the up scale position of the maximum demand pointer,
- -kw values precisely located on the demand scale.

The gears in the Sangamo Type D register are machined to provide proper tooth form which assures consistent and dependable operation. **All** of the gears in this mechanism move vcry slowly with the exception of the pusher arm compound. The pusher arm compound moves up scale slowly and then resets rapidly. Because of this, special attention should be given to the design and material used in the pusher arm compound bearings. To insure long life for the compound, Sangamo uses two ring type sapphire jewels and hardened, polished stainless steel pivots.

Constant friction on the maximum demand pointer is obtained through the use of **a** pressure sensitive silk washer friction system.

Interval timing

The interval timing mechanism can be thought of ,as a clock which establishes the demand interval. This mechanism (Figure **B-5)** consists of the timing motor, reduction gearing and timing cams for triggering the reset operation. The timing cams make one revolution per interval. The length **of** the demand interval **is** determined by the reduction gearing between the motor rotor and the timing cams. This gearing can be designed to allow intervals of a few minutes to several hours.

The heart of the timing mechanism is the motor (Figure *8-6).* It assures that the resetting operation takes place precisely at the end of each time interval. The essential characteristics of a good demand register motor are high torque, stable operation over wide temperature ranges, maintenance of synchronous speed under all conditions, and high insulation level.

> The first Type H motors were used in demand registers in 1948. Since that time, approximately two million have been manufactured. This is the same dependable motor that is used for the precise time requirements of Sangamo time switches and graphic meters.

> **A** polished stainless steel shaft mounted in permanently lubricated porous bronze bearings supports the rotor cup. Operation is not affected by temperatures as low as -50° F or as high as $+250$ °F.

> This motor runs at a synchronous speed of **450** rpm. The 120 *volt* motor will maintain synchronous speed when potential is reduced to as little **as** eight volts.

> The motor coil is encapsulated in an epoxy resin and will withstand surges in excess of 10,000 volts. It is obvious that the motor speed, and thus the timing, **will** not be affected by voltage fluctuations or line surges.

> > EŊ

U.

The Type H is a high torque *(40* mmg.) hysteresis motor. **A** curve representing the torque output with voltage increase and dpcrease is shown in Figure B-7. This **curve** illustrates why the motor stays in synchronism during low voltage conditions and has sufficient torque to overcome any friction that might develop.

Figure *8-6* I **TYPE H MOTOR**

000079 TDM

The motor is mounted below and in front of the demand register. This positioning, together with the inherent characteristics of the motor and the magnetic shield located on polyphase registers between the motor and the watthour meter, insures negligible interference between the motor and potential coil fluxes and makes it possible to calibrate the watthour meter with the demand register detached.

The timing gears are all mounted on stainless steel pivots riding in lifetime lubricated porous bronze bearings. This insures low friction and long life for the register.

The dual timing cams (white, Figure **B-5)** are gear-driven from the motor, assuring accurate interval timing. The drop-off of each of the cams is slightly displaced *so* that the resetting action is properly triggered. The cams are driven through a non-adjustable, lifetime, meta1 **to** metal friction clutch in order *to* allow manual operation. **A** time indicator disk separates the dual timing cams.

Reset operation

Y

li

L

A good reset mechanism in a mechanical register is an absolute necessity as it must perform exactly the same reset operation at the end of every interval. year after year. **A 15** minute interval register will reset approximately 280,000 times in an 8 year service period. It must complete the resetting operation in a minimum length of time and have a minimum number of adjustments.

The operation of the reset mechanism (Figure **B-8)** is triggered by the timing cams. The pusher arm compound is momentarily disengaged from the pusher arm gear wheel, during which time the pusher arm is returned to zero. Immediate reengagement occurs, and the pusher arm begins its up scale movement for the **next** interval.

Figure 8-8 1 **RESET MECHANISM**

The energy for the positive reset operation in the D register is provided by **a** weight and pivot assembly geared to the pusher arm shaft. The resetting operation consists of the following actions (Figure **B-9)** :

- (1st) The rear weighted lever arm drops off the rear timing cam in a downward motion.
- (2nd) This causes the lever assembly to pivot and momentarily lift the driving gear away from the pusher arm compound.
- (3rd) When the pusher arm compound is freed from the driving **gear,** the Counterweight assembly pulls the pusher arm down to the zero position.

Immediately foliowing this action (Figure B-10) :

- **(4th)** The front counterweight lever arm falls off the front timing cam.
- (5th) The resulting gravity action re-engages the pusher arm driving gear with the pusher arm compound.

000081 TDM

ŧ

This completes the reset operation and the pusher arm again moves up scale, counting disk revolutions. Since the pull of gravity on the weights is always the same, the resulting reset action is always the same. There are no springs or clutches that must be adjusted to insure proper resetting.

The original **HM** register incorporated another mechanical approach that is used today. In this approach, the timing train took control of the pusher arm and performed the periodic reset. In Figure B-11, note that in this device, the timing motor released a spring which drove the pusher arm to zero. The objection *to* this arrangement **was** the fact that a clutch was necessary which was strong enough to carry the pusher arm up scale and yet freely permit the down scale drive by the reset spring at the end of each interval. This introduced the mechanical problem of a **clutch** assembly that had to be adjusted between minimum and maximum weak limits, and minimum and maximum strong limits. This was a very difficult adjustment to maintain over a number of years.

Where such an arrangement is used, the serviceman must have a gage to measure this friction and adjust it between these limits. To eliminate such adjustments, Sangamo adopted the gravity reset mechanism which completely disengages gears, resulting in trouble-free dependable operation.

The thermal meter measures average load with an inherent time interval and a response curve which is based on the heating effect of the load rather than on counting disk revolutions during a mechanically-timed interval. While the method of obtaining the maximum average load is entirely different from that of the mechanical demand register. the end result is the same.

The principle upon which the thermal meter operates is the conversion of electrical energy into heat. The heat developed in an electrical circuit of given resistance is proportional to the square of the current (Figure *C-2).*

Figure *C-2*

000083 TDM

 \mathbf{r}

Basic **operation**

A simple analysis of the components **in** a thermal meter can be shown by the following illustrations .

By fusing together two metal strips with different temperature coefficients of expansion, **a** bimetal is formed. These metals will **ex**pand at a different rate when heat is applied, and since they are attached to one another, the strip will bend.

l l l l l l l l l When a bimetal strip is wound into a coil HEATER with the outer end fixed, the inner end can be fastened to a shaft which will rotate with the bending of the bimetal. A pointer can be fitted to the end of the shaft to produce **a** defiection.

ELECTRICAL IN PUT

Figure **C-4**

BIMETAL STRIP --+- *,y* _____._ *--+>*2,* m* ! ' 7'-"7-= _-- - **-__A-**

HEAT

HEATER

NO HEAT-

HEAT APPLIED

The response variation of the bimetal coils in Figures C-4 and *C-5* can be shown graphically in Figure C-0. **An** increase in heat storage capacity lengthens the time that it takes the bimetal coil to reach ultimate deflection, however the response curve is **also** aff ectcd by heat **fiow** rate, which will be explained later.

Figure *C-6*

These basic components and principles are utilized in a thermal meter to measure demand by :

1. Converting heat into **a** heat difference.

SINGLE **BIMETAL** COIL

AFFECTED BY AMBIENT

- 2. Converting a heat difference into **a** temperature difference.
- *3.* Converting **a** temperature difference into a pointer deflection.

Each of these principles will be discussed in detail on the following pages.

Converting heat into a heat difference

Figure C-7

0

Suppose that the bimetal **coil in Figure C-7 is** heated **by** a resistor that has line current passing through it. **As previously** explained, the heat developed **is pro**portional to I'R and the pointer deflection would be proportional to **1;.**

 $\mathcal{L}_{\mathcal{L}}$

The pointer deflection in Figure C-7 would be a function not only of I'R but also of ambient temperature. In effect, a single bimetal coil would act as **a** thermometer. To eliminate the effect of ambient temperature, a second coil that has been carefully matched with the first is put on the shaft so as to turn in the opposite direction (Figure C-8). Now, with a change in ambient temperature, one coil tends to move the pointer up scale and the second tends to move the pointer changes.

Each of these bimetal coils is contained in an enclosure, and one or both of these enclosures may be heated. If only one of the enclosures is heated, the shaft attached to the two coils will turn in proportion to the heat applied.

Utilizing the opposing bimetal coils, which are essential in all thermal meters, the circuit in Figure C-8 constitutes a thermal ammeter.

When current is passed through this meter, heat **(I'R)** is developed in one enclosure causing the temperature **to** rise above that of the other enclosure. **Re**sponding to this difference in temperature, the bimetal coils produce a deflection of the pointer which is proportional to the heat developed and, hence, to the square of the current.

The watt demand meter has a basic circuit as shown in Figure *C-9.* **A** small potential transformer in the meter has its primary connected across the line and its secondary connected in series with two non-inductive heaters of equal resistance, one of which is associated with each enclosure.

O

f

E With *potential only* applied to the meter, a current $\frac{E}{2R}$ circulates through the

heaters as shown in Figure C-10. This circulating current is directly proportional to the line voltage and passes through each one of the heaters. With the potential circuit only energized, heat is developed at the following rates:

'rhus, with only the potential circuit energized, the heat difference is zero, both bimetal coils are at the same temperature, and the pointer does not move.

Now analyze this same circuit with the primary of the potential transformer disconnected and the current section of the circuit compietely energized with line current I (Figure **C-11).** This line current enters the mid-tap of the potential transformer secondary and divides equally. One-half of **I now** passes through each of the two balanced heater elements connected in parallel. With only the current circuit of the meter energized, heat is developed at the following rate:

 \mathbf{t}

Thus with only the current circuit energized. the heat difference is zero, both bimetal coils are at the same temperature, and the pointer does not move. with only the current circuit energized, the heat difference is zero, both brooks are at the same temperature, and the pointer does not move.
With both the current and voltage applied, a current $\frac{E}{2R}$ that is proport

Б

to the line voltage circulates through the heaters as shown in Figure C-12. Current is introduced through the mid-tap in the potential transformer secondary and divides equally. Note that the currents $\frac{E}{2R}$ and $\frac{I}{2}$ add in the rear heater but sub-

tract in the front heater. Heat is developed in the rear heater at a rate:
\n
$$
W_1 = (\frac{E}{2R} + \frac{I}{2})^2 R = \frac{E^2}{4R} + \frac{EI}{2} + \frac{I^2R}{4}
$$

c

 E I_{N/P} E^2 EI I²R tract in the front heater. Heat is developed in the rear heater at a rate:
 $W_1 = (\frac{E}{2R} + \frac{I}{2})^2 R = \frac{E^2}{4R} + \frac{EI}{2} + \frac{I^2R}{4}$
and in the front heater: $W_2 = (\frac{E}{2R} - \frac{I}{2})^2 R = \frac{E^2}{4R} - \frac{EI}{2} + \frac{I^2R}{4}$

Subtracting, the heat difference between the enclosures (and hence the deflection) is:

Because temperature rise in each enclosure is proportional to the heat input, the temperature difference between the two enclosures will correspond to the difference between **these** two values, or simply **E1** (watts).

The current and voltage in the above example were assumed to be instantaneous values, or at unity power factor. To illustrate how **this** meter measures watts at **other** than unity power factor, consider the following example:

The fact that the resistance, R, does not enter into the final equation, EI cos (1, indicates that R may **have** any value. provided the two resistariccs are equal. In practice, it is desirable to choose a value fcr R that will *gi\,e* minimum operating temperatures over the range of operation of the meter. Accordingly. **a value** for R is usually chosen so that $\frac{E}{2D}$ equals half the full load line current. This causes the circulating current to balance half the line current **at** rated load. tinity P.F., and to cause one enclosure to be "cold" under these conditions. 2R

The heaters are made of Nichrome, a nickel chromium afloy, because it has a very high specific resistance. Exact matching of resistance is accomplished by die punching the heater strips at adjacent positions from the same piece of material and pairing off. The resistance **values** arc further checked by actual measurements. To eliminate variations in circuit resistance. the leads are positively attached by solder connections.

Figure C-15 NICHROME HEATERS

The principle of the two-wire thermal meter can be applied to the thermal demand measurement of all singlephase and polyphase services. While the circuit of the two-wire watt demand meter shows **only** one **heater** associated with each **bi**metal coil, *2.* 3, **4** or 6 of these heaters are **utilized** in polyphase meters. Since **any** polyphase circuit consists of two or more two-wire singlephase circuits, the heat inputs of the two-wire circuits can be added **together** and the resultant temperature difference measured with one pair of bimetal coils (Figure C-16).

Figure C-16 POLYPHASE **WATT DEMAND CIRCUIT**

To reduce space requirements and installation costs, on demand billing applications, electric utilities generally specify combination thermal watt demand and kilowatthour nieters. When the meters are so combined. the potential transformer of the thermal watt demand meter can take the form of a secondary winding on the potential electromagnet of the watthour meter. However, Sangamo Lincoln combination meters use separate potential and current transformers to obtain complete indepcndence of the kilowatthour and demand meters. By this method. no sacrifice is made in the excellent operating characteristics of either the **kwh** or demand meter. With separate transformers in all Lincoln watt demand meters, current and voltage values are stepped down to permit compact design and minimum operating temperature.

Converting heat **difference** into **a temperature difference**

Figure C-17 **PRINCIPLE OF HEAT STORAGE**

ATMOSPHERE TEMPERATURE 100' F

When a constant heat is applied to a body which is at the same temperature as its surrounding (Figure C-17a), all the initial heat input is used in raising the temperature of the body. **As** the temperature of the body rises above its original temperature, a portion of the heat is lost to the surroundings due to this temperature difference (Figure C-17b). **As** the temperature of the body becomes higher, a greater proportion of the total heat applied is given off, and less heat is available for raising the temperature of the body (Figure C-17c); until, ultimately, a temperature is reached at which all of the heat applied is given **off** and no further temperature rise takes place (Figure C-17e). This is the "ultimate temperature". The body will remain at this temperature until the heat applied is changed.

If. now, the heat input *to* the body is **stopped,** heat continues momentarily to **flow out** of the body at the same rate as before since the temperature difference between it and the surroundings is the same. However, since heat **is no** longer being applied. the heat which flows out is that which was stored in the body, and accord-

000090 TOM

ingly, the temperature of the body drops rapidly. **As** the temperature drops, the rate of heat loss decreases but the temperature continues to drop until the temperature of the body returns to the temperature of the surroundings.

If a greater volume of water were heated in the kettle of Figure C-17, or if the water had been replaced with a liquid of greater heat storage capacity, the time required to reach ultimate temperature would have been longer, as shown in Figure C-19. The heat storage capacity depends upon the mass and specific **heat** of the materials present, and also to some extent, on their arrangement.

The time required to reach ultimate temperature is also determined by factors that determine the rate of heat flow. Some of these factors are; size, configuration, nature of surfaces, and material. In summary the greater the heat storage capacity, the slower the temperature response (Figure C-19) ; and the greater the heat flow rate, the faster the temperature response (Figure C-20).

The curves of heating or cooling of **a** body under certain ideal conditions are cIosely approximated by the response curves of thermal demand meters. Theoretically, fifteen minute interval Lincoln meters indicate 90% of a steady load fifteen minutes after the load is applied. Further, in the next **15** minutes they respond to *90%* of the remainder of the total load. The deflection reached in any subsequent **15** minute interval **will** always be 90% of the difference between the indication at the beginning of the interval and the steady load (Figure C-21).

R

6

The above response is in accordance with the definition of a *15* minute thermal demand interval as established **by** the "Code for Electricity Meters."

The following indications closely approximate the actual response curves of **15** and 30 minute interval Lincoln **kw** demand meters measuring a steady 100 **kw** load.

By plotting the data from the tables of elapsed time and pusher pointer indication, the following response curves **are** obtained (Figure *C-22).*

Thus far, the response of Lincoln meters has been explained in reference to indication of *a* steady load in the time intcrval of the meter. When extended to chancing loads, the basic principle of thermal demand measurement is that the I..incoln meter responds to *90%* of an increment of load change within **a** tirnc interval (Figure $C-23$).

interval. Note that at the end of the third interval, after the maximum load had been on for only one interval. the maximum demand indication is **8.69 kw or** 96.4% of **the** actual load **(9 kw).** After the load had been on for two intervals, the maxi**mum** demand indication is 99.7% of the actual load. This is, again, because the thermal meter responds to 90% of a change of load during each interval.

000093 TDM

 $\mathbf{1}$

The maximum demand established in Figure **C-23** is typical of obtaining maximum demand under service conditions. In actual practice. maximum demands **are** created by the load increasing from one level to a higher level and usually remaining on the line longer than the duration of a single time interval.

A mechanical type demand meter measures.. demand by averaging the load over a period of time, and equal weight is given to each value of load during that period. With a thermal demand meter, the average is logarithmic and continuous, which means that the more recent the load, the more it is weighted in this average (Figure C-23). **As** time passes, the importance of an instantaneous load value becomes less and less in its effect on the meter indication. The indication of **a** thermal demand meter at any instant depends not only upon the load being measured at that instant but also on the previous values of the load. Because of the inherent time interval, the pointer slowly approaches its ultimate indication.

The first principle dealt with the development of a heat difference through an electrical circuit. This heat difference has now been converted into a temperature difference between the two enclosures to produce **a** response proportional to the load. This response represents a continuous averaging of the load during an inherent interval and *so* constitutes **a** measure **of** demand.

To complete the thermal method of demand measurement, the temperature difference must be converted into pointer deflection.

Converting temperature difference into a pointer deflection

When the two electrical heaters of a thermal kw demand meter create a heat input difference, the two enclosures develop **a** temperature difference proportional to the load. This temperature difference can be measured with bimetal coils which convert the temperature difference into pointer deflection.

The bimetal coils are frequently made of nickel invar and manganese alloy which have **a** substantial difference in temperature coefficients of expansion. The invar is the low expansive metal and manganese is the high. Bimetal strips of correct length (Figure **C-24)** are spot welded to a hardened steel hub and wound into a spiral with the high expansive metal on the inside of the spira1;which causes the coil to expand under a rising temperature. The bimetal is carefully wound **by** a winding machine designed to maintain perfect concentricity and equal spacing **of** each spiral. After winding, the bimetal spiral is heat-treated to set it into a permanent coil. Proper aging of the coil assures permanence and stability **of** its characteristics, providing accurate and reliable performance when in service.

Figure C-24 BIMETAL STRIPS AND COIL

HIGH EXPANSIVE METAL ON INSIDE

COIL IN **CUT AWAY ENCLOSURE**

BIMETAL COILS ON **SHAFT Figure C-25**

Before the coils are placed on **a** shaft, they are carefully matched. This matching process assures that coils which have identical expansion characteristics are placed on the same shaft. If they were not exactly matched, a change in ambient temperature would cause a slight pointer deflection. The coils are placed on a steel shaft in opposition to each other (Figure C-25). The hub in the center of the spiral coils is firmly positioned on a knurled portion of the shaft.

The heater strips and bimetal coils are exactly positioned in the center of enclosures (Figure C-26). The outer end of the coil is riveted to a retaining ring so that when a temperature difference occurs, the shaft will rotate due to the expansion or contraction of the coil. These enclosures **are** small and compact, and are mainly composed of a thermal insulating material, bakelite. Nichrome heaters are located on the inside periphery of the enclosures and are held in place by **a** retaining ring. **A** steel ring and **washers** assure uniform distribution of heat to **the** bimetal coils and add heat storage capacity. The complete assembly **is the** thermal measuring element.

MEASURING ELEMENT AND POINTERS

Figure C-27

When the thermal measuring element is placed in a meter, the shaft with a pusher pointer attached, turns on polished stainless steel pivots. Advancing torque is developed by the rear (driving) coil and opposing torque by the front (retarding) coil, so that the rotation of the shaft, and hence the deflection of the pointer, is proportional to the load being measured. **A** sun shield placed over the measuring element (Figure C-28) assures that direct rays of the sun will not produce an ambient temperature difference between the coils.

Figure C-28 EXPLODED VIEW OF COMPLETE SINGLEPHASE THERMAL METER

Pusher pointer deflection, which fluctuates according to the magnitude of the kw load, is read on a linear scale.

To indicate the maximum up scale deflection, the maximurn demand pointer *is* advanced by the pusher pointer. The pusher pointer has a small tab which contacts the maximum demand pointer during up scale movement. The tab is twisted so that only an edge surface contacts the maximum demand pointer **to** prevent **a friction** band between the two pointers.

The maximum demand pointer remains at the highest up scale, or **kw** value, reached by the pusher pointer until manually reset at the end of the billing period. It is carefully balanced and held in place with a silicone-controlled clutch (Figure C-29) when not being advanced up scale by the pusher pointer. The specially prepared silicone compound provides high holding torque for the maximum demand pointer against vibration or rapid motion. **How**ever, the clutch exerts negligible restrain**ing** torque to the slow steady movement **of** the pusher pointer.

The scale

The thermal demand scale covers an **arc** of 70 degrees. It is over **4** inches in length and is nearly horizontal for easy reading. Lincoln combination meters are designed *so* that demand pointers do not obstruct reading of the kilowatthour dials.

1~1111 scale values and the number of divisions for **a** given full scale value are determined by industry standards as listed below.

'Values per division for 100 division scales will depend on the capacity of the demand meter to which the scale **is** applied. Multiplying constants for demand meters having IOOdivirion scales will be based on the full scale values listed above, (full **scale** value), **in** oddition to the ratio of instrument transformers when applicable. **IO0**

Figure C-30 ASSORTED THERMAL DEMAND SCALES

The above **full** scale **values** when applied to one and two stator meters are expressed as **"class"** on the meter nameplate. "Class" denotes the maximum load range **in** amperes of the demand meter.

Standard full scale values for various classes of one and two stator watthourthermal watt demand meters are listed on **the** following page.

d-

!

CLASS DESIGNATIONS--FULL SCALE VALUE

Figure C-31 I MULlTlPLlER DESIGNATIONS

a $\frac{3}{5}6\frac{\pi}{6}634\frac{3}{4}636\frac{\pi}{6}64$

KILOWATTS $\frac{20}{40}$ 30 40 50 60 10 80

Multiplying constant

When a multiplying constant is applicable to the **kwh** reading only, it is adjacent to the **kwh** dials and is preceded by the words "Kilowatt Hours -Multiply By" as in Figure C-31a.

When a multiplying constant is applicable to the thermal demand reading only, it is adjacent to the **kw** scale and is preceded by the **words** "Multiply Demand By" as in Figure C-31b

When the same multiplying constant is applicable to both the demand reading and the **kwh** dials and is determined by the ratio **of** the instrument transformers with which the meter is used, a removable plate having the **words** "Multi**ply** Both Readings **By-.-" is** attached to the demand scale as in Figure c-31c.

 φ

Dual range

Because thermal metcrs are also rated in per cent of full scale, the full *scale* rating of the demand meter should be matched as closely to the load **as** possible. Since it is frequently necessary to increase the meter capacity with load growth, a dual range feature is advantageous.

Lincoln singlephase and polyphase thermal combination meters are available in dual range construction. This makes it possible to use **the** meter on **a** low range initially and to double the range later. Full scale values are determined by the connection of the mid-tapped current transformer secondary (Figure *C-32).* On a singlephase meter (Figure C-33) the range is changed by transferring **a** single lead on the terminal block of the base plate. On polyphase meters (Figure **C-34),** the change consists of moving the transformer secondary links at the top of *the* meter. The terminal blocks are marked to indicate the range on which the demand meter **is** connected.

The demand scales must also be changed to correspond with the selected range. By merely removing the two screws holding them in place, the two scales may be interchanged. When a two range meter **is** supplied, the demand scales are located ond above the other with the **low** range on top. The utility companyidentification **tag** is not disturbed by this operation.

SINGLEPHASE DUAL RANGE **CHANGEOVER Fiaure C-33** *d*

.,I. *n.* **D,.**

SINGLEPHASE DUAL RANGE CIRCUIT DIAGRAM Figure C-32

POLYPHASE DUAL RANGE CHANGEOVER Figure C-34

Accuracy

Lincoln thermal demand meters are calibrated for **1%** full scale accuracy at all points on the demand scale. **As** in any indicating instrument, per cent of accuracy is expressed in relation to full scale or full load value.

For example, a Lincoln meter with 96 kw full scale capacity would have ± 0.96 maximum error limits. At one-half scale, this meter would still have \pm 0.96 maximum error limits but, because the load value is then 48 kw, the accuracy would be $\pm 2\%$ of load.

At lower scale points, the error in percentage of load will increase inversely with the load (Figure C-35). Consequently, it is advisable to use demand meters as well as any other indicating meter in the upper one-half of their scales.

SCALE ACCURACY

The excellent field accuracy of thermal meters is, in part, a result of the simplicity of design (only one moving part). Careful selection, matching, and aging of the bimetal coils are other factors of prime importance. Compensation for fluctuations of ambient temperatures (sun shield, enclosure design, deflection adjustment) give stable accuracy in all installations.

Adjustments

000100 TDM

On Lincoln combination demand meters, there are only two adjustments affecting the demand meter. Both are easily accessible from the front of the meter and are made with a screwdriver.

The zero adjustment consists of a light, spiral hairspring attached to the shaft of the pusher pointer (Figure C-36). Rotating the zero adjustment screw adds to or subtracts from the deflection position of the pusher pointer. It affects the pointer deflection equally all along the scale and its action is independent **of** the pusher pointer position. For example, if *it is* adjusted to reduce the no load (zero) position by one per cent, it also reduces the full load indication by one per cent. For this reason, the zero adjustment should always be made before the deflection adjustment.

DEFLECTION ADJUSTMENT DRAWING Figure **C-37** ^I

The deflection adjustment consists of a spring attached to the pusher pointer (Figure C-37). The amount of torque exerted on the pusher pointer is proportional to the scale position of the pointer. At zero indication no torque is exerted ; but as the load increases, torque is exerted approximately proportional to the deflection position. For example, if the deflection adjustment is operated to reduce full scale indication by two per cent, then the indication at half load **is** reduced by two per cent of the reading, or one per cent of full scale.

Q1

'Testing and maintenance

Periodic tests of Lincoln demand meters can coincide with the normal test periods of kilowatthour meters. This sustained accuracy **is** the result of the inherent design and the fact that thermal meters have only one moving part.

Thermal meters can be tested in either the shop or the field. However, for economical reasons the most practical method is to gang-test them at a central location. This is because of the time involved in producing actual loading conditions for pusher pointer deflection. Deflection of the pusher pointer cannot be checked by mechanical means.

When gang-testing is employed (Figure **C-38).** the meters should be left in the test room and on potential for a minimum of two hours. The current circuit is disconnected and the meter covers should be in place, simulating no load conditions. After this warming period on potential only, the zero adjustment of the pusher pointer can be made (Figure **C-39).**

Full scale indication can be checked during gang-testing by two different methods. One **is** to use a graphic or indicating me'ter which has been carefully calibrated as the standard meter. The meters to be tested are connected in series with the standard meter and a load of $3/4$ scale or higher applied. After three intervals. the Taximum indications are compared. **A** second method is to control the load at the above level by means of an automatic load holder. Here again adjustment is made after three intervals (Figure **C-40).**

Figure C-38 (THERMAL METERS ON GANG-TEST BOARD

Figure *C-39* I **ZERO ADJUSTMENT**

Figure C-40 | DEFLECTION ADJUSTMENT

 $:SMOI$ when large groups are gang-tested. These held tests can be accomplished as folalso be used in shop testing, but the testing cost per meter is not as economical as Other methods can be used to field test themal meters. These methods can

- ragnibasa basare readings. meter under test and allow it to remain long enough to compare the maxi-1. Put a specially-calibrated meter of the same capacity in series with the
- The load must be held steady during the third time interval. 2. Use an indicating wattmeter and controlled phantom or resistance load.
- and it should closely correspond to the indication of the pusher pointer. standard at frequent intervals. The average load can then be calculated 3. With a stop watch, that the speed of the kwh meter dish or a rotating

the steady load. has been applied, should not be multiplied by $\frac{9}{10}$ and checked with the value of minute interval meter, the pusher pointer indication 15 minutes after a steady load one interval should be used before accuracy is checked. For example, with a 15 When testing from a no load conlition, it is recommended that more than

matched during manufacture and do not require further attention. formed before they are assembled in the meter. The heating elements are precisely bimetal coils will remain stable indefinitely because of the aging processes perless steel pivots, no lubrication is required on any part do the thermal meter. The Since the only moving part, the bimetal shaft, moves slowly on polished stain-

iacturer. renewed according to factory instructions with compound specified by the manudoes not hands a sufficient annount of silicone compound; it should be cleaned and meter is subjected to external vibration or shock (tapping the cover). It the clutch points. Sufficient holdiory torque is present if the pointer will not move when the mum demand pointer is correct when the pointer remains stationary at all check mum demand pointer to one or more positions on the scale. Balance of the maxicheeked during periodic tests. Clutch action can be checked by moving the maxiunspection during the life of the meter. The action of the clutch should be The maximum demand pointer clutch is the only component that requires

QUeStiOghS and answers

The previous sections have explained the theory of operation and the construction of mechanical demand registers and thermal demand meters. There are **a** number of questions that arise regarding a direct comparison of the two instruments. This section is intended to give frank answers to some of the most fre $quently$ encountered questions.

1. HOW CAN THE POPULARITY OF SINGLEPHASE THERMAL DEMAND METERS BE EXPLAINED?

ANSWER: Sales of singlephase thermal demand meters are approximately 3 to 1 over mechanical demand registers because there is little or no price differential,

and the thermal demand meter offers:

- 1. Service reliability and minimum maintenance (only one moving part).
- 2. Sustained accuracy to permit the demand portion to be tested only when the watthour meter would normally be tested.
- *3.* Dual Range feature for load growth.

of the mechanical demand registers, and because :

2. HOW CAN THE POPULARITY OF POLYPHASE MECHANICAL DEMAND REGISTERS BE EXPLAINED?

ANSWER: Sales of polyphase mechanical demand registers are approximately 3 to **1** over thermal demand meters because there is **a** slight price advantage in favor

1. The "theoretical" accuracy advantage is considered worthwhile by some utilities **for** large loads.

2. Periodic installation tests are required on polyphase services more **fre**quently than on singlephase services and the mechanical demand register **I** easier to test in the field.

4

3. IS THERE A DIFFERENCE IN INDICATION OF THE TWO **INSTRUMENTS FOR SHORT DURATION LOADS?**

ANSWER: Yes, as shown in the response curves of Figure D-1, the thermal meter will read higher for loads of a duration less than approximately 80% of the rated time interval. Some utilities feel that the higher reading for short interval loads is justified. This special case would rarely occur at the time of maximum demand.

Figure *D-i* 1 MECHANICAL **AND** THERMAL RESPONSE CURVES

4. WHAT IS PEAK SPLITTING AND HOW DOES IT AFFECT THE INDICATION OF THER-MAL AND MECHANICAL DEMAND INSTRUMENTS?

ANSWER: Peak splitting may best be defined by the following illustration (Figure D-2):

«

It can be seen from the above drawing that the load was applied when the mechanical demand register was half way through the reset cycle of the first interval. **The** load was then removed **at** the middle **of** the second interval and as **a** result, the demand indicated was only one half of the maximum load. The thermal demand **meter** is not affected by peak splitting because its interval starts when a change of load is applied. Therefore, in the above example, the thermal demand meter indicates **90%** of the load, while the mechanical demand register indicates **50%** due to peak splitting.

5. DOES PEAK SPLITTING INTRODUCE AN ERROR IN **THE CUSTOMER'S MAXIMUM DEMAND?**

- ANSWER: No. it does not. During the average billing period of one month there are *2.880* fifteen minute demand intervals. The likelihood that all the load changes will occur in the middle of an interval or that the maximum load will remain on for less than 15 minutes is extremely improbable. Extensive field tests by utilities verify the fact that both thermal and mechanical demand meters indicate the average maximum load during the billing period.
- **6. ARE SPECIAL DEMAND METERING PRACTICES USED ON CUSTOMERS WHO VARY THEIR LOADS TO TAKE ADVANTAGE OF PEAK SPLITTING?**
	- ANSWER: There are two schools of thought on this question. Some utilities feel that the customer can control his load in any manner and that he shouldn't be billed for any more than the demand meter indicates. Those utilities who want to bill on the actual maximum demand during any billing period can try altering the reset time of a mechanical demand register at the beginning of each billing period and also remove the interval pointer. One foolproof method of preventing a customer from taking advantage of peak splitting is to use a thermal demand meter, since it continuously averages the load and does not reset.
- **7. RECENT ADVERTISEMENTS HAVE SUGGESTED THAT THERMAL DEMAND METERS ARE LESS ACCURATE THAN MECHANICAL DEMAND REGISTERS SINCE THEY ONLY INDICATE** *90"o* **OF THE AVERAGE DEMAND DURING THE TIME INTERVAL. IS THIS TRUE?**
	- ANSWER: No! The inherent response has little or nothing to do with the accuracy of demand billing. In order to make the mechanical demand register accuracy look better than the thermal demand meter accuracy, the following curves have been shown (Figure **D-3):**

Figure D-3 I **A SPECIAL CASE** TO **FAVOR MECHANICAL DEMAND REGISTERS**

Obviously, the "other type" is the thermal demand meter. This is **a** *very special case* and does not present a factual comparison of the two **meters.** Those who use the above illustration certainly must realize that the **mathe**matical possibility of such a condition occurring is **as** remote as the possibility of having the reset operation occur in the middle of the interval. (Figure **D-4).**

This second very special case would show the thermal demand meter indicating 90% at the end of the first interval, while the mechanical demand register is indicating only 50% . This is also an unfair comparison. In actual practice neither of these special cases occurs continually during the billing period. For an objective comparison of the two meters, the following actual conditions must be considered:

- 1. The maximum load is nearly always on the line longer than the duration of a single demand interval.
- 2. The maximum load is reached a number of times during the billing period.
- 3. The maximum load is not suddenly thrown on the line. Maximum demands are created by the load increasing from one level to a higher level. The two instruments can be more objectively compared by the following example (Figure D-5) where the load increases from 90 to 100 kw and remains at that level for more than one interval.

♦

伪

RESPONSE TO ACTUAL LOAD CONDITIONS

During the first 15 miuntes the maximum demand pointer of the mechanical demand register could be indicating any value from 90 to 100 kw. The value would depend on when the pusher arm reset. However, since the load was applied for at least 30 minutes (two intervals), the maximum demand pointer would indicate 100 **kw** a: **the** end of the second interval.

#

d

The thermal demand meter pusher pointer and maximum demand pointer were indicating 90 **kw** when the load **was** increased to 100 kw. Since the thermal demand meter inherently responds to *90%* of the change in load during an interval, the pusher pointer would be indicating 99 **kw** at the end of the first **15** minutes. During the next 15 minutes the thermal demand meter would respond to *90%* of the difference between the indication and the maximum load. Consequently, the thermal demand meter would he indicating 99.9 kw at the end of 30 minutes.

Since it is impossible *to* read an indicating instrument of this type any closer than $1/10$ of 1% , the mechanical demand register and thermal demand meter are indicating the same value or the average maximum load. From this example, it can be seen that either instrument will accurately measure the maximum average load during a billing period.

- a. **IT HAS BEEN FREQUENTLY STATED THAT THE ACCURACY OF THE MECHANICAL DEMAND REGISTER IS COMPARABLE WITH THE WATTHOUR METER ITSELF SINCE THE DEMAND INDICATION IS OBTAINED BY DIRECT GEAR DRIVE FROM THE METER DISK SPINDLE. 15 THIS TRUE?**
	- ANSWER: Not altogether. **As** previously explained the accuracy of these instruments is expressed in percent of full scale. Even if the watthour meter is in perfect calibration, the error of a mechanical demand register which has 1% full scale accuracy can be $\pm 4\%$ of the load at quarter scale. Although the mechanical demand register does count disk revolutions, it is not correct to state that the accuracy is due to this direct gearing. The rated accuracy of a mechanical demand register is determined by the eccentricity of the pointer hub with respect to the arc of the scale, the time required for resetting, and possible backlash in the gearing. Sangamo's precisely manufactured D registers are rated at $\pm 1\%$ of full scale.

9. **HOW DO THE TWO INSTRUMENTS COMPARE WITH REGARD TO READING ERRORS?**

ANSWER: The mechanical demand register has a scale length approximately twice that

of a thermal demand meter, and accordingIy may be read more closely. **As** an example, a **24** kw mechanical demand register would have 120 divisions of 0.2 kw per division, whereas a **24** kw thermal demand meter wouId have **48** divisions of *0.5* kw per division.

On the other hand, the thermal demand scale covers a smaller arc and is nearly horizontal for easy reading. Lincoln combination meters have the demand scale so located that the demand pointers do not interfere with the reading of the kilowatthour dials.

10. IS THE MECHANICAL DEMAND REGISTER MORE ACCURATE THAN THE THERMAL DEMAND' METER?

ANSWER: This question can be more easily answered **by** considering both the "theoretical accuracy" and the "field accuracy" of mechanical demand registers and retical accuracy" and the "field accuracy" of mechanical demand registers and the Lincoln registers. The Sangamo Type D register and the Lincoln

Ø

凒

thermal meter are factory calibrated to 1% of full scale. However, there is a slight advantage in favor of the mechanical demand register in considering the theoretical accuracy of the instruments when both have been carefully designed and are functioning properly. When the watthour meter is correctly calibrated, it is easier to maintain the accuracy of the mechanical dcrnand register under laboratory conditions. The difference in the theoretical accuracy of the two types of demand meters is a matter of a **few** tenths of a percent. The human elements involved in reading make it very difficult to verify this slight difference in theoretical accuracy.

Field accuracy on the other hand concerns the overall accuracy in billing the customer. This billing, of course, must be accomplished under all climatic conditions for extended periods of time. Each of these instruments has its advantages and disadvantages for producing long range accuracy. Some of these advantages and disadvantages are pointed out elsewhere, but the net result is that *the all important "field accuracy"* **of** *the two instruments is* coni*parable.*

11. **WHAT ARE THE FACTORS THAT AFFECT THE** LONG **RANGE "FIELD ACCURACY" OF MECHANICAL DEMAND REGISTERS AND THERMAL DEMAND METERS?**

ANSWER: 1. The failure rate (see question 17) has a great effect on the average accuracy of demand billing.

- 2. Too little holding torque in the maximum demand pointer of either instrument would result in the possibility of errors from vibration.
- 3. Power outages can affect the timing of polyphase mechanical demand registers. If there **is** a temporary failure of one of the three phases serving a customer, and the timing motor is connected across that phase, the timing motor stops and the pusher arm continues up scale until the motor is reenergized and calls **for** resetting. With an outage of this type, the demand indication will be high or **off** scale. This cannot happen with a thermal demand meter since it continually averages the load and is independent of the watthour meter.
- **4.** Direct sun and temperature changes can cause an error in a thermal demand meter if the bimetal coils are not carefully matched. They must also be shielded *so* that there is no ambient temperature gradient between the two bimetals. Lincoln combination demand meters are designed with a metal shield as **a** part of the register. For this reason the Lincoln meter can utilize a clear glass cover and avoid the possibility of paint flaking from a painted cover.

A considerable amount of error can be introduced into the demand register **if** the timing motor has not been designed to withstand extremes in temperature without slowing down (and giving a high demand).

- *5.* Reading errors limit the "field accuracy" of both instruments.
- 6. The maximum demand indication of both instruments must be above half scale to take advantage of **the** accuracy ratings.

Long range performance and sustained accuracy **of** both mechanical demand registers and thermal demand meters depend upon the quality built into them through design, engineering, and care in the manufacture.

I 2. **IS THERE AN ERROR INTRODUCED BY THE TIME REQUIRED FOR RESETTING OF** *c-* **THE MECHANICAL DEMAND REGISTER?** \'. *\9* ;&,&,A:\$'

ANSWER: 'The time required to reset the D register is only 1.5 to *4.5* scconds for **a** 15 minute interval rcgisrer and this minor error is included in the overall rating of $\pm 1.0\%$ full scale for the instrument.

HOW DOES TESTING TIME COMPARE FOR THE TWO TYPES OF INSTRUMENTS?

ANSWER: This question should be answered on the basis of field testing and shop testing, and will depend to a great extent on the methods employed by the tester.

> In general, less time is required to test a mechanical demand register in the field. It is usually only necessary to utilize a register checker, or to run the register **up** scale knowing the pusher arm ratio, and then to observe the resetting operation after the meter has been reinstalled. It is also advisable to check the speed of the timing motor.

I

a

The time required for field test of the thermal demand meter varies greatly because of the different approaches to testing of these meters. One possible scheme where **a** customer's load is steady or where a steady phantom load can be used is to count the disk revolutions of the watthour meter (or to observe a portable wattmeter). This observation of the load should be continued for approximately 15 minutes (or the interval of the meter). The scale reading can then be checked. **A** comparison method utilizing a standard thermal demand meter would require more time since the meters must reach full load indication.
Shop testing is usually accomplished through gang-testing because it is

more economical than testing either type of instrument individually. When tested individually, the mechanical demand register does require less time **for** calibration ; however, when gang-testing is employed there is relatively little difference in the time required for the two instruments. The test equipment and procedures can vary in complexity with different companies and therefore it is difficult to compare testing time. If the two instruments are to be gang-tested, the cost of the equipment required is approximately the same.

14. CAN DEMAND METERS BE SAMPLE TESTED AS RECEIVED FROM THE MANUFAC-TU RER?

ANSWER: Yes. One large utility with approximately *50,000* demand meters in service **has** found that it can rely on sampling **10%** of the singlephase and polyphase Lincoln demand meters received.

15- DOES THE TYPE OF DEMAND INSTRUMENT AFFECT THE DETERMINATION OF PERIODIC TEST INTERVALS?

ANSWER: Periodic test intervals for demand instruments are determined primarily by utility commissions and individual utility practices. The type of customer **and** size of load are more important considerations in determining periodic **test** intervals than the type of demand instrument.

> Thermal demand meters are sometimes assigned longer periodic test in**tervals,** but this may be due. in part, to **the** fact that they are more frequently used on singlephase services.

Either demand instrument can fit into the trend toward longer periodic test intervals-especially if an inspection is scheduled between tests.

16. HOW FREQUENTLY SHOULD MECHANICAL AND THERMAL GRAPHIC DEMAND METERS BE TESTED?

ANSWER : **As** explained in **g15.** the test interval depends on type of service. size of load, and regulations. Because they are employed on larger loads, graphic demand meters are usually tested more frequently than indicating demand meters. Checks can be made on these meters in between periodic test as follows: The mechanical graphic demand meters can employ a counter as a check against the kwhr register on the number of contacts received during the billing period.

> The chart of a thermal graphic demand meter can be integrated with a radial planimeter and the calculated kwhr value compared with the kwhrs indicated on the integrating meter. This can be done quite accurately (within $\pm 2\%$) when the load has not rapidly fluctuated.

17. WHAT FAILURE RATE AND MAINTENANCE COST HAS BEEN EXPERIENCED ON THERMAL AND MECHANICAL DEMAND INSTRUMENTS?

ANSWER : Utility experience indicates that maintenance costs will be much lower on thermal demand meters. One company with over 30,000 demand meters in service, including both demand registers and thermal demand meters, reported that the failure rate of Lincoln demand meters was only *0.3%* per year compared with 1.3%, 4.0% and 12.5% per year on various makes and models of demand registers. Obviously, a failure of either demand meter results in an error of many times the accuracy ratings of the instruments.

> Cost of a complete overhaul and test of the two instruments is approximately the same, but experience indicates that because of the inherent design there is less likelihood of a thermal demand meter requiring a complete overhaul.

18. IT HAS BEEN STATED THAT THE MECHANICAL DEMAND REGISTER HAS AN AD-VANTAGE OVER THE THERMAL DEMAND METER IN THAT IT "FAILS SAFE". IS THIS TRUE?

ANSWER: It is true that in many instances a failure of the motor in a mechanical demand register will result in a "zero" or **"off** scale" reading. However, failure of other components, such **as** reset mechanism, gearing and maximum demand pointer, to operate properly does not necessarily cause the demand register to "fail safe." It is also possible for some timing motors to lose synchronism and run slower than synchronous speed (older types could run faster), causing a higher reading because of the longer interval. This slowdown is highly improbable with the H motor because of its inherent high torque.

> **A** thermal meter could also be described as failing safe, in that if lightning were to knock out **a** potential or current transformer, **a** very large error would result. This would enable the billing department to recognize the discrepancy between the total kilowatthours consumed and the erroneous demand indicated. On the other hand, with the mechanical demand register, if potential element of the meter fails, both kilowatthours and the demand indication will be **off** and the billing department might interpret this as a change in customer load.

> > 0001 10 TDM

19. WHAT IS THE EXPECTED LIFE OF A DEMAND REGISTER MOTOR?

ANSWER: One manufacturer recently answered this question in the following way: "Experience indicates that the expected life of the timing motor is usually about ten years. The motor is easily removed by loosening two mounting screws and disconnecting two leads."

> On the other hand, the Type H motor, which is the heart of Sangamo's Type **D** register, doesn't even require relubrication for a period of at least ten years, and its life expectancy is comparable to that of the watthour meter.

20. ARE THERE ANY SAVINGS IN INVENTORY THROUGH THE USE OF MECHANICAL DEMAND REGISTERS?

ANSWER: Yes, but this depends to a great extent on the standards of individual utilities. Universal demand registers which offer interchangeability between meters of different service ratings would reduce inventory needs. Universal registers would be more popular except for the problem of handling odd multipliers in the meter and billing departments.

21. WHAT BURDEN IS IMPOSED ON INSTRUMENT TRANSFORMERS BY THESE TWO INSTRUMENTS?

ANSWER: Obviously, the demand register itself imposes no extra burden on a current transformer. The additional burden on one phase of a potential transformer from the Type H Demand Register motor is 4.2 **V.A.** and 2.3 watts for all voltage ratings.

> Because Lincoln combination meters include independent thermal meter elements and transformers, an additional burden is imposed on the instrument transformer. On a class 10, **2.5** amps, 120 V 2-wire singlephase kwh meter, at rated load, burden in the current coil circuit is 0.70 V.A. and 0.26 watt, while potential burden is **7.5 V.A.** and 1.0 watt. On the same rated combination kwh-kw Lincoln meter current burden is 0.86 **V.A.** and 0.60 watt, while potential burden is 8.9 V.A. and **3.1** watts. The small additional burden in **a** combination meter is well within the burden characteristics of miniature current and potential transformers. Combination thermal meters can be used along with other measuring instruments without overtaxing the instrument transformer.

#

 $\ddot{}$

22. CAN BOTH MECHANICAL AND THERMAL INSTRUMENTS BE UTILIZED FOR MEAS-UREMENT OF KYAR AND KVA DEMAND?

ANSWER: Thermal demand meters have been utilized for direct measurement of kvar and kva demand for many years. The meters are completely self-contained for both singlephase and polyphase measurement of these values. Lincoln meters of this type offer independent measurement of demand and energy in one package.

> Kva and kvar demand measurement can also be obtained through mechanical demand registers, although many manufacturers' watthour meters **re**quire an external phase shifting transformer. However, Sangamo has designed self-contained polyphase kva-hour and kvar-hour meters which eliminate external phase shift transformers. Type D mechanical demand registers **or** contact operated instruments can be used with these meters to measure **kva** and kvar demand.

When a mechanical demand register installation requires either kva or kvar demand *and kwh measurement*, a separate meter must be used to obtain the kwh. The same installation can bc metered with one combination thermal meter, which provides both kva or kvar demand *and* **kwh** measurement.

23. WHAT FEATURES OF **BOTH INSTRUMENTS PERMIT MEASUREMENT OF GROWING LOADS WITHOUT CHANGING OUT THE COMPLETE METER?**

Thermal demand meters are available with a dual range feature that is easily changed in the field. This feature provides extra demand measurement capacity and better accuracy by keeping the demand reading in the upper half of the scale as loads increase. Again, recalibration is not necessary on Lincoln meters with a range change.

The time to make the change for either type of instrument is approximately the same.

24. IT HAS BEEN STATED THAT THE DRIVING TORQUE IS HIGHEST AT TIME OF **MAXIMUM LOAD FOR A MECHANICAL DEMAND REGISTER, AND THAT IT** IS **LOWEST AT TIME OF MAXIMUM LOAD FOR A THERMAL DEMAND METER.** IS **THIS TRUE?**

ANSWER: No. This statement is apparently the result of confusing the driving torque of a meter with the torque required to move the demand pointer system. The driving torque of *either* type of demand meter is maximum at the time of times greater than the torque required to move the demand pointer. maximum demand; and, in a properly designed meter of *either* type, is many

25. HOW CAN A METER DEPARTMENT DETERMINE THE MOST DESIRABLE DEMAND INSTRUMENT FOR THESE PARTICULAR APPLICATIONS?

ANSWER: Consideration should **be** given to the applicable rate, test schedules, operating problems, and operating practices which all affect the economics of meter selection.

> Although these questions and answers have dealt primarily with the comparison between mechanical and thermal demand instruments, an evaluation of two makes **of** the same basic type of instrument might reveal as many variations *as* a comparison **of** two basic types of instruments (thermal and mechanical).

0001 12 TDM

SANGAMO ELECTRIC COMPANY

 $\boldsymbol{\lambda}$

SPRINGFIELD, ILLINOIS

Jim DeMars 09/24/02 04:43 PM

To: Magda RothrnanlCSIFPL@FPL, Cathy CarpenterlCSIFPL@FPL cc: Subject: kWh vs. kWd Billing

Okay! After much research and discussion here is the "official" Meter Engineering response to the question:

"Can a demand meter register zero kWh but register kWd?"

If the meter is a thermal demand meter (first character "1" or **"4"),** the customer should not be billed on kWd if the kWh is zero.

The reasons are:

- *0* The meter could just be off-zero.
- *0* If potential is applied to the meter and there is no current flow, thermal meters have demonstrated the ability to register a little demand due to thermal heating from direct sunlight.

If the meter is electronic (first character "6", "D",), the customer should be billed on the kWd even though there does not appear to be kWh.

The reason is:

Transformer-rated meters can have large multiplier. The smallest kWh value that can be recorded is the value of the transformer constant. Say the multiplier is 240, when the kWh reading indicated 00001, this would correspond to a billed value of 240 kWh. But the demand register has the resolution of two decimal places and it would show some value long before the kWh reached the value of "I".

We are in the process of retiring the thermal demand meters but with a population of 91,000, it is going to take a few years. When we do, then one rule will fit all applications.

,

4

FPL-ESE TEST REPORT SUMMARY

FPL-ESE TEST REPORTS KWD FULL SCALE % ERROR COMPARISON

Chart1

7/12/2004

FPI-FOR CONPARISON

FPL-ESE TEST REPORTS KWD TEST POINT % ERROR COMPARISON

driawiae iNeporting Oerv
Miami ∞ Orlando ∞ Tampa

 $\begin{array}{c} \mathbb{W}{\bf{or}}{\bf{ldwide}} \ \mathbb{Re} {\bf{porting}} \ S{\bf{ervice}}\\ {\bf{Miami}} \ \approx \ \text{Orlando} \ \approx \text{Tampa} \end{array}$

Worldwide Reporting Service
Miami ~ Orlando ~ Tampa

 $\overline{}$

 $\mathcal{E}^{\mathcal{E}}$

Worldwide Reporting Service
Miami & Orlando & Tampa

Bromley-Complainant-Direct *72*

1

11

12

13

14

15

2 3 4 5 6 7 8 9 10 MR. MOYLE: Cochran, let me give you sort of a little synopsis of why we felt the need to have you here, and what Ken and I think we've managed to work out, but you're aware, I believe, about the independent test issue that occurred over in Bradenton, and then the disparity between some numbers, in an effort to have additional testing performed in Miami, right?

MR. KEATING: As I understand, just to make sure I got this right, the disparity is between some independent test results done in Bradenton, versus test results done at FPL's facility in Miami?

16 17 18 19 20 21 22 23 24 MR. MOYLE: Right. And anyway, I was asking some questions, and the one question I asked was, if FP&L had taken additional steps to investigate why the disparity in numbers between the Miami facility and the Bradenton facility, and what those steps were, that sort of line of questioning. Mr. Bromley thought it might be impinging upon privilege.

25 He asked for a break, I gave him a

Worldwide Reporting Service
Miami & Orlando & Tampa

————————————————————

Worldwide Reporting Service
Miami & Orlando & Tampa

Worldwide Reporting Service
Miami ∞ Orlando ∞ Tampa